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NASA CONTRACTOR REPORT 166416 ✓

(NASA-CR-166416) NUMERICAL CALCULATION OF
THE PARAMETERS OF THE EFFLUX FROM A HELIUM
DEWAR USED FOR COOLING OF HEAT SHIELDS IN A
SATELLITE Final Report, Sep. 1981 - Jul.
1982 (Illinois Univ., Urbana-Champaign.)

N83-11463

Unclass
G3/34 00810

NUMERICAL CALCULATION OF THE PARAMETERS OF
THE EFFLUX FROM A HELIUM DEWAR USED FOR
COOLING OF HEAT SHEILDS IN A SATELLITE

BY

✓ Keith Brendley and John C. Chato
Department of Mechanical and Industrial Engineering
University of Illinois at Urbana-Champaign

July 1982



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**National Aeronautics and
Space Administration**

**Ames Research Center
Moffett Field, California 94035**

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ABSTRACT

The parameters of the efflux from a helium dewar in space were numerically calculated. The flow was modeled as a one-dimensional, compressible, ideal gas with variable properties. The primary boundary conditions are flow with friction and flow with heat transfer and friction. Two PASCAL programs were developed to calculate the efflux parameters: EFFLUXD and EFFLUXM. EFFLUXD calculates the minimum mass flow for the given shield temperatures and shield heat inputs. It then calculates the pipe lengths, diameter, and fluid parameters which satisfy all boundary conditions. Since the diameter returned by EFFLUXD is only rarely of nominal size, EFFLUXM calculates the mass flow and shield heat exchange for given pipe lengths, diameter, and shield temperatures.

NOMENCLATURE

A	area
B	function of M ($= [1 + (\gamma - 1)/2] M^2$)
C_p	specific heat
D	diameter
f	friction factor
h	fluid film constant
k_f	fluid thermal conductivity
L	pipe length
M	Mach number
m	mass flow
Nu	Nusselt number
P	pressure
P_0	stagnation pressure
Pr	Prandtl number
Q_w	shield heat exchange
q	heat flux
Re_D	Reynolds number
St	Stanton number
T	temperature
T_0	stagnation temperature
T_w	wall (shield) temperature
T_{aw}	adiabatic wall temperature
V	velocity
x	distance

Greek Letters

γ	specific heat ratio
ϵ	shield effectiveness
ρ	density
ρ_0	stagnation density
μ	viscosity

Subscripts

1	entrance
2	exit

1. INTRODUCTION

One of the primary objectives in the design of a long lifetime helium dewar is to minimize the parasitic heat load. This is accomplished to a large extent through low-conduction supports and vapor-cooled shields imbedded in multi-layered insulation (MLI). The vapor-cooled shields can be modeled as a series of heat exchangers with constant temperature walls and constant heat input to the helium efflux. With the ambient pressure of space being approximately zero, the fluid flow is compressible with a Mach number of unity at the exit throat. The solution of this model basically consists of determining (1) the length of pipe required to provide the necessary heat transfer and (2) the diameter required for the optimum mass flow to reach the Mach number of unity at the throat.

This solution entailed the development of two PASCAL programs: EFFLUXM and EFFLUXD. EFFLUXD is used to optimize mass flow and define the length and diameter of the heat exchanger pipe. Since this diameter is only rarely and by chance of nominal size, EFFLUXM is used to determine the mass flow and heat absorption of a similar heat exchanger but with a nominal diameter close to theoretical diameter determined through EFFLUXD. These programs are explained in APPENDIX A.

Although the original scope of the project was to develop these programs with the number of shields, the temperature of these shields and their position as given, a second-law thermal analysis of the shield system would seem beneficial to the project as a whole. Preliminary calculations indicate that the optimal number of shields is less than the five previously cited by other sources. Information on this subject will be forwarded at a later date.

2. THEORETICAL CONSIDERATIONS

The shield heat exchanger system is a bit unusual in that both shield temperature and heat input are specified at each shield from an overall heat transfer analysis of the entire insulation system developed elsewhere. If a shield heat exchanger system is analyzed with the effectiveness method, it becomes clear that for a given effectiveness the mass flow is a dependent variable. This mass flow may be minimized as an explicit function of shield temperature, heat input, and effectiveness. As an example of the method, the three-shield system shown in Fig. 1 will be analyzed. The initial stagnation temperature, the shield temperatures, the shield heat inputs, and the shield effectiveness (or shield-to-helium exit temperature difference) are all given. From the definition of effectiveness as shown in Eq. (1), the fluid stagnation temperatures at points 1, 2, and 3 may be calculated.

$$\epsilon = (T_{o \text{ out}} - T_{o \text{ in}})/(T_w - T_{o \text{ in}}). \quad (1)$$

Note that the stagnation temperature at points 1 and 2 are outlet conditions for one shield and inlet conditions for the next. A single mass flow, however, will generally create different effectiveness in each shield. Consequently, the optimal mass flow must be found which will provide the maximum effectiveness in one shield. To accomplish this objective, each j th mass flow is calculated as a function of the initial stagnation temperature, the j th wall temperature, and the summation of heat inputs to the j th wall.

$$\dot{m}_j = [\sum_{i=1}^j Q_{wi}]/[C_p(T_{oj} - T_{o \text{ in}})] \quad (2)$$

The maximum resulting mass flow is the optimal mass flow for the given conditions. With this mass flow, the actual shield effectivenesses can be calculated. For constant fluid properties, these effectivenesses would

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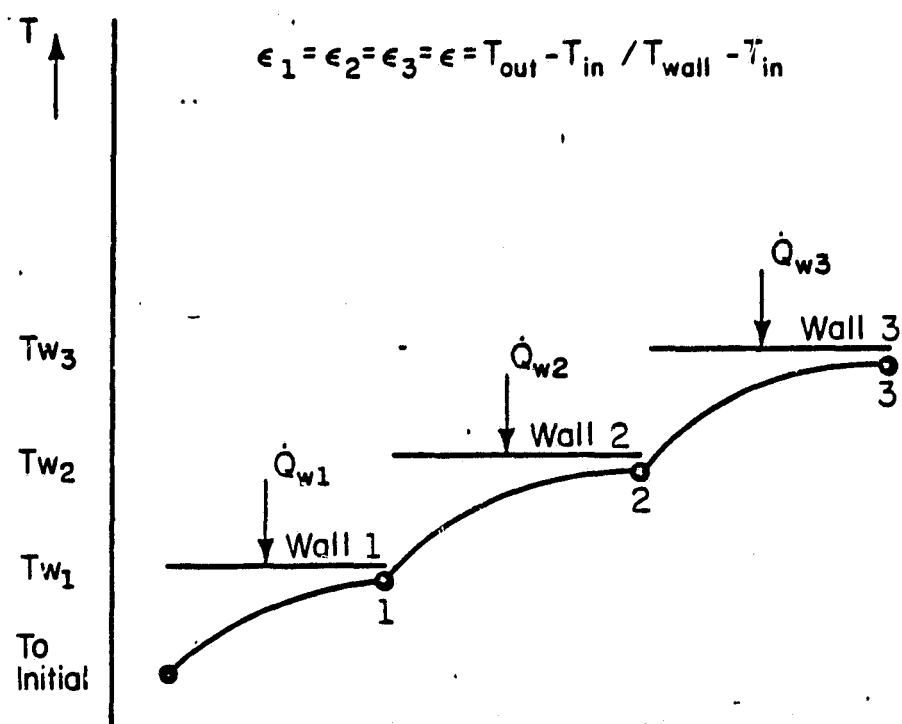


Figure 1 Temperature profile of helium pipe flow over three radiation shields.

directly define the heat exchanger lengths. The widely varying properties of the helium flow, however, force the sectional solution used in the EFFLUXD program.

In addition to the problem of variable fluid properties, the flow is compressible with a Mach number of 1 at the exit throat. Since the mass flow and length have already been set, the condition of $M = 1$ at the exit must be satisfied by determining the proper diameter. This is done implicitly in EFFLUXD as shown in APPENDIX A. If a nominal rather than calculated diameter is used, the mass flow is no longer optimized and must be varied until the exit condition of $M = 1$ is reached. This is done implicitly in EFFLUXM as shown in APPENDIX A.

Both of these programs use a number of functions and procedures which solve for the exit Mach number of a short section of pipe, given the inlet Mach number and the boundary conditions. The first of these procedures calculates the pipe entrance Mach number as a function of mass flow and diameter as explained in APPENDIX B.

For sections with friction only (connecting pipes between shields or valves), the flow is assumed laminar below a Reynolds number of 2000 and turbulent above that. This transition changes the frictional and thermal behavior of the fluid. In laminar flow, the friction factor is a function of Reynolds number only and the Nusselt number is constant; in turbulent flow, both the friction factor and Nusselt number are functions of Reynolds and Prandtl numbers as illustrated more clearly in APPENDIX C.

Frictional effects are also important in a heat transfer section of sufficient length. Before the exit Mach number of a short section may be calculated, the exit stagnation temperature must be known. This can be calculated with the simple energy balance in APPENDIX D. One interesting

aspect of this balance is that it is a function of the adiabatic wall temperature. For laminar flow, this is equivalent to the stagnation temperature; for turbulent flow, the adiabatic wall temperature lies between the fluid stagnation temperature and the actual fluid temperature. However, for subsonic pipe flow, the adiabatic wall temperature can be assumed to equal the stagnation temperature with no appreciable error.

After the exit stagnation temperature is calculated, the governing differential equation for compressible flow with heat transfer and friction may be simplified using Reynolds analogy and the previous energy balance as described in APPENDIX E. The resulting equation is solved as shown in APPENDIX F with a fourth-order Runge-Kutta method.

Within each small section, the fluid properties may be assumed constant, but overall thermal conductivity, viscosity, and Prandtl number vary as a function of temperature. The Prandtl number, as a function of absolute temperature, was modeled with a fifth-order least squares fit as shown in Fig. 2, while the thermal conductivity and viscosity are modeled with third-order fits as shown in Figs. 3 and 4. The temperatures used are the section inlet fluid temperatures.

The diameter, once determined, remains constant. Any constriction of the diameter in beginning sections increases the diameter in later sections to maintain the optimal mass flow. Furthermore, diameter increases would either require the design and manufacture of smooth venturi sections or would result in pressure loss across the area change. This added pressure loss would again require increasing the diameter even more or require increasing the mass flow. Earlier suggestions of diameter increases seemed to arise from the problem of extremely long pipes; heat transfer sections about two orders of magnitude longer than needed were proposed since, at the time, the heat

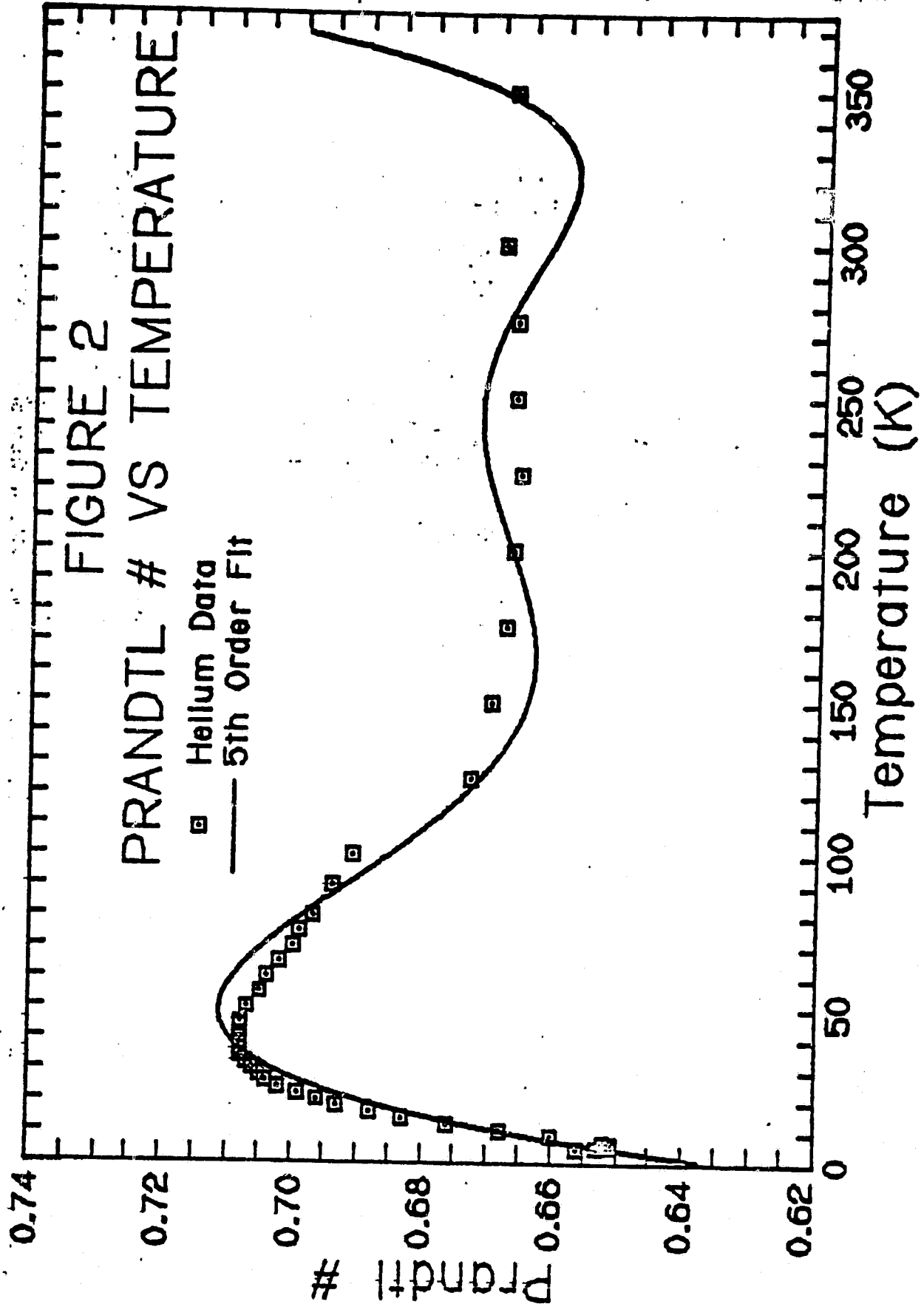


Figure 2 Prandtl Number versus Temperature

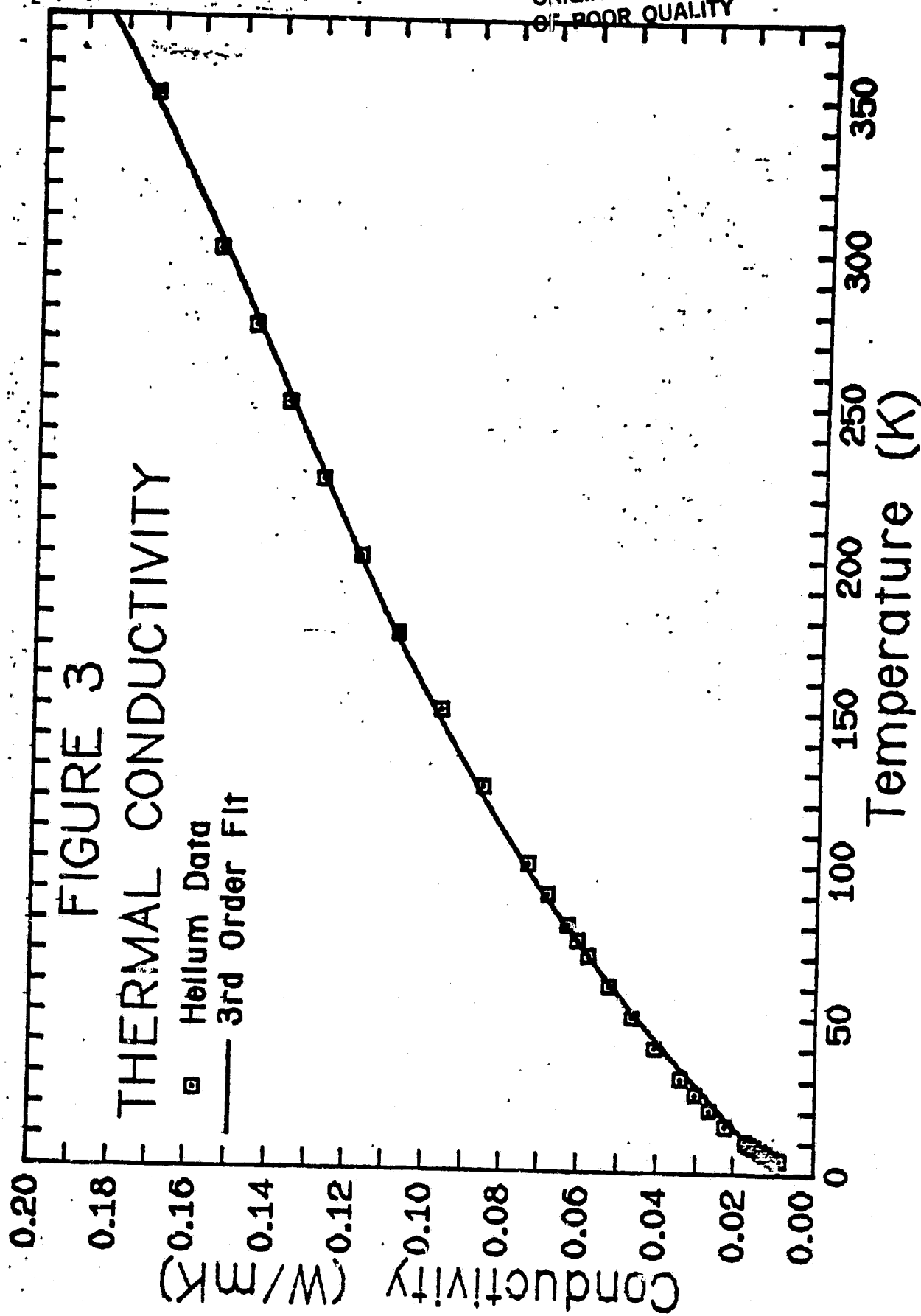


Figure 3 Thermal Conductivity

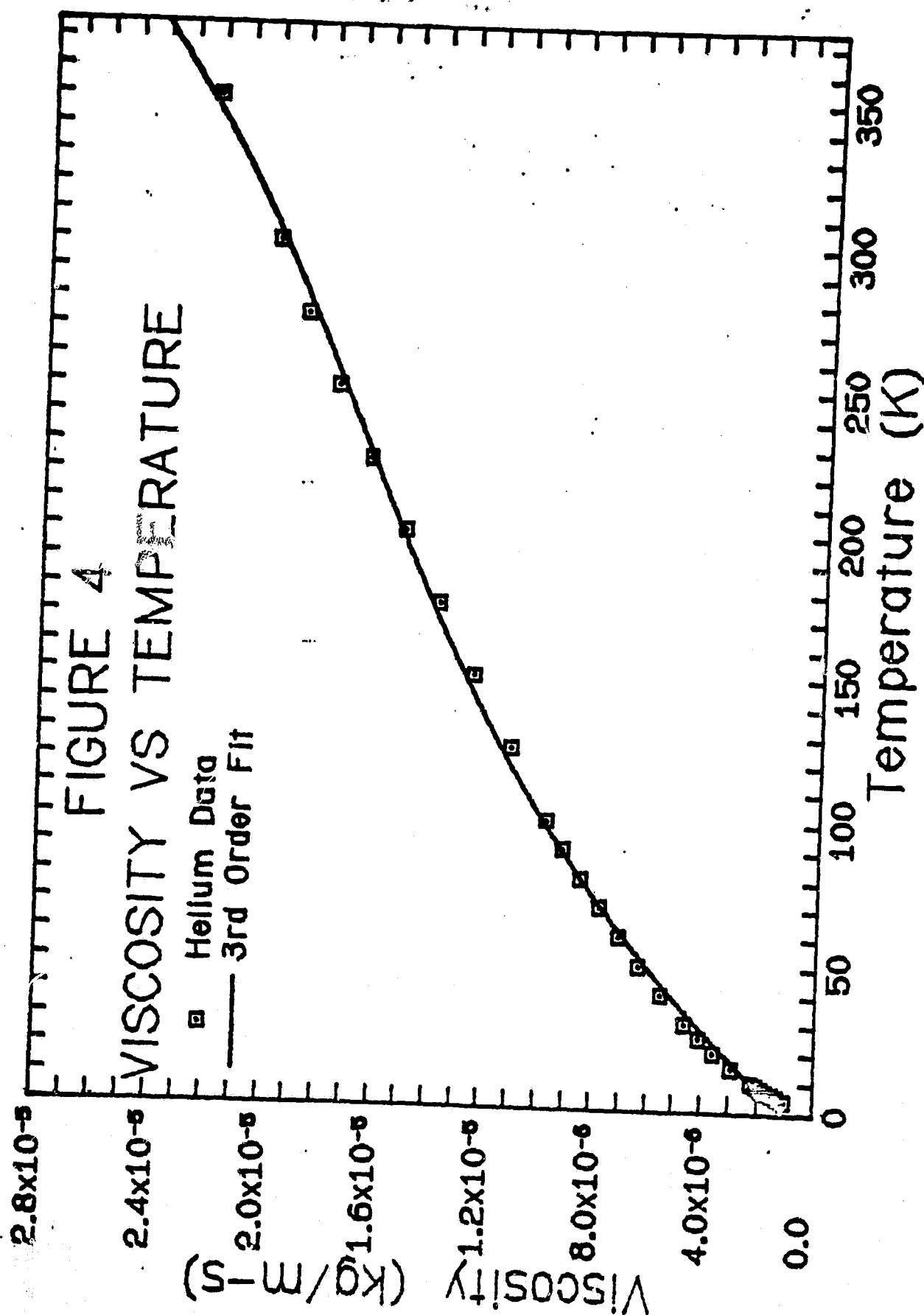


Figure 4 Viscosity versus Temperature

transfer mechanism was not fully understood. Even though it is difficult to see the advantage of area changes, a procedure to deal with them is in EFFLUXD as shown in APPENDIX H. This approach, however, was never utilized.

To summarize the design procedures, the following components can be identified:

1. The sections in contact with heat shields are treated as heat exchangers with both heat transfer and friction influencing the flow of gas.
2. The connecting sections between heat shields are considered as adiabatic lengths with friction only.
3. Valves in the system can be treated as adiabatic sections as in point 2, with equivalent lengths adjusted to yield the proper pressure drop for a given flow rate and entrance conditions. Depending on the data available for the valve, this calculation will require iterations by the user of the program.

APPENDIX A

The inputs and outputs of the PASCAL programs EFFLUXD and EFFLUXM should be self-explanatory. The units are consistently SI. Example runs of both programs with a five-shield system follows. The input data used were taken from "Feasibility Study for Long Lifetime Helium Dewar," Parmley, R. T., Lockheed Missiles and Space Company, Inc., Contract No. NAS2-10848, NASA, 1981, pp. 3-11.


```

VAR
NUMWALLS, NUMDIV, SPACECNT, N, NTEST: INTEGER;
I, LOOPCOUNT, PKEY, TAB, PIPECNT, WALLCNT: INTEGER;
EFFOUTD, POUTD: TEXT;
EFFIND1: REALOUT;
EFFIND2: INTOUT;
P, T, RHO, DIA: DIVISIONS;
X, QY, M, PO, TO, Q, RE, F, NU: DIVISIONS;
QV, TV, LTEMP, LSPACE, EFF, QWALL: COUNT;

```

L,LTOTAL,LOWBOUND,HISOUND,TOSAVE:REAL;
MDOT,MAXEFF,DIA1,DIVLENGTH:REAL;
M1,T01,P01,T1,P1,RHO1,F1,RE1,Q1,QX1,MU1:REAL;

```

PROCEDURE INDATA(INDEX:INTEGER);
  BEGIN (* INTERACTIVE INPUT *)
    IF INDEX > 0 THEN
      BEGIN
        (* READ IN DATA *)
        WRITE('INPUT DATA AS PROMPTED BY TERMINAL');
        WRITE(' ');
        WRITE('WHEN ASKED FOR A SERIES OF DATA SUCH AS THE SHIELD');
        WRITE('TEMPERATURES, INPUT THE DATA IN THE ORDER FROM DEWAR');
        WRITE('TO THE OUTER BOUNDARY. ');
        WRITE(' ');
        WRITE('ENTER THE NUMBER OF RADIATION SHIELDS');
        READLN;
        READ(NUMWALLS);
        WRITE(EFFIND1,NUMWALLS);
        WRITE('NUMBER OF SHIELDS=',NUMWALLS:5);
        WRITE(' ');
        WRITE('ENTER SINGLE DIVISION LENGTH (M)');
        READLN;
        READ(DIVLENGTH);
        WRITE(EFFIND1,DIVLENGTH);
        WRITE('DIVISION LENGTH (M)=',DIVLENGTH:15:5);
        WRITE(' ');
        WRITE('ENTER PIPE LENGTHS NOT ON SHIELDS (M)');
        FOR I:=1 TO NUMWALLS+1 DO
          BEGIN
            READLN;
            READ(LSPACE(I));
            WRITE(EFFIND1,LSPACE(I));
            END;(*FOR*)
        WRITE('ENTER TEMPERATURES OF RADIATION SHIELDS (K)');
        FOR I:=1 TO NUMWALLS DO
          BEGIN
            READLN;
            READ(TV(I));
            WRITE(EFFIND1,TV(I));
            END;(*FOR*)
        WRITE('ENTER HEAT INPUTS OF RADIATION SHIELDS (W)');
        FOR I:=1 TO NUMWALLS DO
          BEGIN
            READLN;
            READ(QV(I));
            WRITE(EFFIND1,QV(I));
            END;(*FOR*)
        WRITE('ENTER MAXIMUM HEAT EXCHANGER EFFECTIVENESS AS A FRACTION');
        READLN;
        READ(MAXEFF);
        WRITE(EFFIND1,MAXEFF);
        WRITE('SHIELD EFFECTIVENESS=',MAXEFF:15:7);
        WRITE(' ');
        WRITE('ENTER INITIAL STAGNATION T (K)');
        READLN;
        READ(T0(I));
        WRITE(EFFIND1,T0(I));
        TV(0):=T0(I);
        WRITE('INITIAL STAGNATION TEMPERATURE (K)=',T0(I):15:5);
        WRITE(' ');
        WRITE('ENTER INITIAL STAGNATION P (PA)');
        READLN;
        READ(P0(I));
        WRITE(EFFIND1,P0(I));
        WRITE('INITIAL STAGNATION PRESSURE (PA)=',P0(I):15:5);
        WRITE(' ');
        WRITE('ENTER NUMBER OF STEPS IN RUNGE-KUTTA METHOD');
        READLN;
        READ(N);
        WRITE(EFFIND1,N);
        WRITE(' ');
        WRITE('STEPS IN RUNGE-KUTTA=',N:5);
        END;(*IF*)
      ELSE
        (* READ FILE *)
        BEGIN
          RESET(EFFIND1);

```

```

RESET(EFFIND2);
READ(EFFIND2,NUMWALLS);
READ(EFFIND1,DIVLENGTH);
FOR I:=1 TO NUMWALLS+1 DO READ(EFFIND1,LSPACE(I));
FOR I:=1 TO NUMWALLS DO READ(EFFIND1,TW(I));
FOR I:=1 TO NUMWALLS DO READ(EFFIND1,QW(I));
READ(EFFIND1,MAXEFF);
READ(EFFIND1,T0(I));
TW(0):=T0(1);
READ(EFFIND1,P0(I));
READ(EFFIND2,N);
END('ELSE')
END;(* INDATA *)

```

PROCEDURE OUTDATA;

```

BEGIN
  WRITE(EFFOUTD,'LAWFLOW OUTPUT':60);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'VALUES OF INPUT':60);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'INLET STAGNATION TEMPERATURE (K)':50,T0(1):15:10);
  WRITE(EFFOUTD,'INLET STAGNATION PRESSURE (PA)':50,P0(1):15:10);
  WRITE(EFFOUTD,'NUMBER OF RADIATION SHIELDS':50,NUMWALLS:15);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'TEMPERATURES OF RADIATION SHIELDS':50);
  FOR I:=1 TO NUMWALLS DO WRITE(EFFOUTD,'SHIELD TEMPERATURE (K)':30,I:2,'=':1,TW(I):15:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'HEAT INPUTS TO RADIATION SHIELDS':50);
  FOR I:=1 TO NUMWALLS DO WRITE(EFFOUTD,'SHIELD HEAT (W)':30,I:2,'=':1,QW(I):15:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'PIPE LENGTHS BETWEEN SHIELDS':50);
  FOR I:=1 TO NUMWALLS+1 DO WRITE(EFFOUTD,'SPACE LENGTH (M)':30,I:2,'=':1,LSPACE(I):15:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'SOLUTION SECTION SIZE (M)':50,DIVLENGTH:15:10);
  WRITE(EFFOUTD,'MAXIMUM SHIELD EXCHANGER EFFECTIVENESS':50,MAXEFF:15:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'VALUES OF OUTPUT':60);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'MASS FLOW (KG/S)':50,MDOT:20:10);
  WRITE(EFFOUTD,'PIPE DIAMETER (M)':50,DIA(LOOPCOUNT):20:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'TOTAL PIPE LENGTH (M)':50,LTOTAL:20:10);
  WRITE(EFFOUTD,'SHIELD LENGTHS':40);
  FOR I:=1 TO NUMWALLS DO WRITE(EFFOUTD,'SHIELD LENGTH (M)':30,I:2,'=':1,LTEMP(I):15:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'TOTAL HEAT TRANSFER (W)':50,QWALL(NUMWALLS+1):20:10);
  WRITE(EFFOUTD,'SHIELD HEAT TRANSFER':40);
  FOR I:=1 TO NUMWALLS DO WRITE(EFFOUTD,'SHIELD HEAT (W)':30,I:2,'=':1,QWALL(I):15:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'OVERALL EXCHANGER EFFECTIVENESS':50,EFFNUMWALLS+1:10:10);
  WRITE(EFFOUTD,'SHIELD EFFECTIVENESS':40);
  FOR I:=1 TO NUMWALLS DO WRITE(EFFOUTD,'SHIELD EFF':30,I:2,'=':1,EFF(I):15:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'EXIT STAGNATION TEMPERATURE (K)':50,T0(NUMDIV):20:10);
  WRITE(EFFOUTD,'EXIT STAGNATION PRESSURE (PA)':50,P0(NUMDIV):20:10);
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,' ');
  WRITE(EFFOUTD,'FOR AN OUTPUT OF SECTION FLOW VARIABLES, INPUT 1. OTHERWISE INPUT 0.');


READLN;



READ(PKEY);



IF PKEY = 1 THEN



  BEGIN



    WRITE(EFFOUTD,'LENGTH':7,'MACH NUMBER':14,'STAG TEMP':14,'STAG PRES':14);



    WRITE(EFFOUTD,'TEMP':10,'PRESSURE':10,'NUSSLT':10,'REYNOLD NUM':15);



    WRITE(EFFOUTD,'SECTION HEAT':15,'TOTAL HEAT':15);



    WRITE(EFFOUTD,' ');



    FOR I:=1 TO NUMDIV DO



      BEGIN



        WRITE(EFFOUTD,X(I):8:5,' ':4,M(I):10:8,' ':4,T0(I):10:4);



        WRITE(EFFOUTD,' ':3,P0(I):10:8,' ':2,T(I):8:4,' ':2,P(I):8:3);



        WRITE(EFFOUTD,' ':2,MU(I):8:3,' ':5,RE(I):10:5,' ':5,Q(I):10:7);


```

```

WRITELN(EFFOUTD, ' :5,QX(13:10:7);
END(*FOR*)
END(*IF*)
WRITELN('FOR A PLOT FILE ENTER 1, OTHERWISE ENTER 0');
READLN;READ(TAB);
IF TAB = 0 THEN
  BEGIN
    WRITELN('FOR PLOT OF STAGNATION TEMPERATURE ENTER 1. ');
    WRITELN('FOR PLOT OF STAGNATION PRESSURE ENTER 2. ');
    WRITELN('FOR PLOT OF HEAT TRANSFER ENTER 3. ');
    READLN;READ(TAB);
    WRITELN(POUTD,NUMDIV);
    FOR I:=1 TO NUMDIV DO
      BEGIN
        CASE TAB OF
          1: WRITELN(POUTD,T0(I));
          2: WRITELN(POUTD,P0(I));
          3: WRITELN(POUTD,QX(I));
        END(*CASE*)
      END(*FOR*)
    FOR I:=1 TO NUMDIV DO WRITELN(POUTD,X(I));
  END(*IF*)
  WRITELN('NUMBER OF INTERACTIONS=':50,LOOPCOUNT:5);
  WRITELN('FOR INTERACTION INFORMATION, INPUT 1. OTHERWISE INPUT 0');
  READLN;
  READ(PKEY);
  IF PKEY = 1 THEN
    FOR I:=1 TO LOOPCOUNT DO
      WRITELN(EFFOUTD,'INTERACTION=':20,I:3,'DIAMETER=':20,DIAC(13:15:10);
    END(* OUTDATA *)
  
```

```

FUNCTION FWR(N,P:REAL):REAL;
(* SOLVES FOR N=P *)
VAR A:REAL;
BEGIN
  A:=P*LN(N);
  FWR:=EXP(A);
END(* FWR *)

```

```

FUNCTION B(M:REAL):REAL;
(* SOLVES OFTEN OCCURRING FUNCTION *)
BEGIN
  B:=1.0+(K-1.0)*SQR(M)/2.0;
END(* B *)

```

```

FUNCTION TEMP(T0,M:REAL):REAL;
(* CALCULATES LOCAL TEMPERATURE *)
BEGIN
  TEMP:=T0/B(M);
END(* TEMP *)

```

```

FUNCTION PRES(P0,M:REAL):REAL;
(* CALCULATES LOCAL PRESSURE *)
VAR A,C:REAL;
BEGIN
  C:=B(M);
  A:=K/(1-K);
  PRES:=P0*PWR(C,A);
END(* PRES *)

```

```

FUNCTION DENSE(P,T:REAL):REAL;
(* CALCULATES LOCAL DENSITY *)
BEGIN
  DENSE:=P/(R*T);
END(* DENSE *)

```

```

FUNCTION MU(T:REAL):REAL;
VAR A,B,C,D:REAL;
BEGIN(*MU*)
  A:=1.10419245531E-04;
  B:=1.14677594448E-07*T;
  C:=3.222732924042E-10*SQR(T);

```

```
D:=4.72570029/922E-13*T*SQR(T);
MU:=(A+B+C+D);
END;(*MU*)
```

```
FUNCTION COMD(T:REAL):REAL;
  VAR A,B,C,D:REAL;
  BEGIN(*CONDUCTIVITY CALCULATION*)
    A:=0.0157218071;
    B:=6.3416325E-04*T;
    C:=4.99407521E-07*SQR(T);
    D:=4.52803094E-10*T*SQR(T);
    COMD:=(A+B+C+D);
  END;(*COMD*)
```

```
FUNCTION PRANDTL(T:REAL):REAL;
  VAR A,B,C,D,E,F:REAL;
  BEGIN(*PRANDTL NUMBER CALCULATION*)
    A:=0.570232994;
    B:=0.0055872291*T;
    C:=7.58324532E-05*SQR(T);
    D:=4.32720501E-07*T*SQR(T);
    E:=1.11661445E-09*SQR(T)*SQR(T);
    F:=1.07403842E-12*T*SQR(T)*SQR(T);
    PRANDTL:=(A+B+C+D+E+F);
  END;(*PRANDTL*)
```

```
FUNCTION MOUT(M1,M2,D1,D2:REAL):REAL;
  (* USED IN CASE 1 OF NEWTON *)
  VAR A,C:REAL;
  BEGIN
    A:=(K+1.0)/(2.0*(K-1.0));
    C:=B(M2)/B(M1);
    MOUT:=(M1/M2)*PWR(C,A)-SQR(D2/D1);
  END;(* MOUT *)
```

```
FUNCTION DMOUT(M1,M2:REAL):REAL;
  (* USED IN CASE 1 OF NEWTON *)
  VAR A,C:REAL;
  BEGIN
    A:=B(M2)/B(M1);
    C:=(K+1.0)/(2.0*(K-1.0));
    DMOUT:=(((K+1.0)/2.0)*(B(M1)/B(M2))-1.0/SQR(M2))*M1*PWR(A,C);
  END;(* DMOUT *)
```

```
FUNCTION MFRIC(M1,M2,D,F,L:REAL):REAL;
  VAR MFR:REAL;
  (* USED IN CASE 2 OF NEWTON *)
  BEGIN
    MFR:=(SQR(M2)-SQR(M1))/(K*SQR(M1*M2)+(K+1.0)/(2.0*K)*LN(B(M2)*SQR(M1)/(B(M1)*SQR(M2)))-4.0*F*L/D);
    MFRIC:=MFR;
  END;(* MFRIC *)
```

```
FUNCTION DMFRIC(M1,M2:REAL):REAL;
  (* USED IN CASE 2 OF NEWTON *)
  VAR DMF:REAL;
  BEGIN
    DMF:=2.0/(K*M2*SQR(M2)+(K+1.0)/K*M2*SQR(B(M1))/(PWR(M1,4)*PWR(B(M2),3)));
    DMFRIC:=DMF;
    IF DMF < 1.274 THEN (*FUNCTION UNSTABLE, MACH=1 APPROACHED*)
      DMFRIC:=1.0;
    END;(* DMFRIC *)
```

```
FUNCTION MFRST(M,D,MDOT,P0,T0:REAL):REAL;
  (* USED IN CASE 3 OF NEWTON *)
  VAR A,C:REAL;
  BEGIN
    A:=B(M);
    C:=(K+1.0)/(2.0*(1.0-K));
    MFRST:=(M*P0*PI*SQR(D)/4.0)*SQR(K/(R*T0))*PWR(A,C)-MDOT;
```

END; (* MFRST *)

```

FUNCTION DMFRST(M,D,P0,T0:REAL):REAL;
(* USED IN CASE 3 OF NEWTON *)
VAR A,C,E:REAL;
BEGIN
  A:=M(N);
  C:=(K+1.0)/(2.0*(1.0-K));
  E:=(3.0*K-1.0)/(2.0*(1.0-K));
  DMFRST:=(PI*SQR(D)/4.0)*P0*SQRT(K/(R*T0))*(PWR(A,C)-SQRT(M)*(K+1.0)/2.0*PWR(A,E));
END; (* DMFRST *)

```

```

FUNCTION NEWTON(KEY:INTEGER; M1,D,F,L,LTOT,P0,T0:REAL):REAL;
VAR MNEW,MOLD,DIFF,D1,D2,A,MDOT,DMF,MF:REAL;
COUNT:INTEGER;
BEGIN (* NEWTON *)
  DIFF:=1.0;
  CASE KEY OF
    1: BEGIN
      (* CALCULATES EXIT MACH NUMBER FOR SECTION WITH AREA CHANGE *)
      D1:=D;
      D2:=F;
      MOLD:=SQRT(D1/D2)*M1;
      WHILE (DIFF) > 0.001 DO
        BEGIN
          MNEW:=MOLD-MOUT(M1,MOLD,D1,D2)/(DMOUT(M1,MOLD));
          DIFF:=ABS(MOLD-MNEW);
          MOLD:=MNEW;
        END (* WHILE *)
      END; (* 1 *)
    2: BEGIN
      (* CALCULATES EXIT MACH FOR SECTION WITH FRICTION ONLY *)
      MOLD:=1.01*M1;
      COUNT:=1;
      WHILE (DIFF) > 0.00005 AND (COUNT < 10) DO
        BEGIN
          DMF:=DMFRIC(M1,MOLD);
          MF:=MFRIC(M1,MOLD,D,F,L);
          MNEW:=MOLD-MFRIC(M1,MOLD,D,F,L)/DMF;
          DIFF:=ABS(MOLD-MNEW);
          IF DMF = 1.0 THEN (*FUNCTION UNSTABLE, EXIT LOOP*)
            BEGIN
              DIFF:=0.00001;
              MNEW:=1.0;
              END; (* IF *)
          COUNT:=COUNT+1;
          MOLD:=MNEW;
        END (* WHILE *)
      END; (* CASE 2 *)
    3: BEGIN
      (* CALCULATES INLET MACH NUMBER *)
      MDOT:=L;
      MOLD:=M1;
      WHILE (DIFF) > 0.00001 DO
        BEGIN
          MNEW:=MOLD-MFRST(MOLD,D,MDOT,P0,T0)/DMFRST(MOLD,D,P0,T0);
          DIFF:=ABS(MOLD-MNEW);
          MOLD:=MNEW;
        END (* WHILE *)
      END; (* 3 *)
    END; (* CASE *)
  NEWTON:=MOLD;
END; (* NEWTON *)

```

```

PROCEDURE INLETMACH(D,MDOT,P0,T0:REAL; VAR M,T,P,RHO:REAL);
(* CALCULATES INLET CONDITIONS *)
VAR A:REAL;
BEGIN (* INLETMACH *)
  M:=NEWTON(3,M,D,0,MDOT,0.0,P0,T0);
  T:=TEMP(T0,M);
  P:=PRES(P0,M);
  RHO:=DENSE(P,T);
END; (* INLETMACH *)

```

```

PROCEDURE AREACHANGE(D1,D2,T0,P0:REAL;VAR M,T,P,RHO,RE,F,MU,PR:REAL);
(* FOR DIAMETER CHANGE *)
VAR A:REAL;
BEGIN
M:=NEWTON(1,M,D1,D2,0,0,0,0);
T:=TEMP(T0,M);
P:=PRES(P0,M);
RHO:=DENSE(P,T);
RE:=RE*D1/D2;
IF RE <= 2000 THEN
  BEGIN (* LAMINAR FLOW *)
    F:=16.0/RE;
    MU:=3.66;
    END(*IF*)
ELSE
  BEGIN (* TURBULENT FLOW *)
    F:=0.046/PWR(RE,0.2);
    MU:=0.023*PWR(RE,0.8)*PWR(PR,0.4);
    END(*ELSE*)
END(*AREACHANGE*)

```

```

PROCEDURE FRICTION(T0,DIA,F,L:REAL;VAR M,T,P,RHO,P0,Q:REAL);
(* FOR FLOW WITH FRICTION ONLY *)
VAR A,C,NEWM:REAL;
BEGIN
NEWM:=NEWTON(2,M,DIA,F,L,LTOTAL,0,0);
IF NEWM > 1.0 THEN NEWM:=1.0;
A:=8*(NEWM)/R(M);
C:=(K+1.0)/(2.0*(K-1.0));
P0:=P0*M/NEWM*PWR(A,C);
M:=NEWM;
T:=TEMP(T0,M);
P:=PRES(P0,M);
RHO:=DENSE(P,T);
Q:=0;
END(*FRICTION*)

```

```

FUNCTION DIAMETER(MDOT,T0,P0,M:REAL):REAL;
(* CALCULATES DIAMETER GIVEN MDOT AND MACH *)
VAR A,C:REAL;
BEGIN(*DIAMETER*)
A:=4.0*MDOT*SQRT(R*T0/K)/(PI*M*P0);
C:=(K+1.0)/(2.0*(K-1.0));
DIAMETER:=SQRT(A*PWR(R(M),C));
END(*DIAMETER*)

```

```

PROCEDURE SECTION(NUMTOT:INTEGER; LENGTH,DIVL:REAL; VAR I:DIVISIONS; VAR NUMDIV:INTEGER);
(* DIVIDES PIPE LENGTHS INTO SMALLER SECTIONS *)
VAR
  RESL,REALDIV:REAL;
  I,J:INTEGER;
BEGIN(*SECTION*)
IF NUMTOT = 1 THEN
  BEGIN
    NUMTOT:=0;
    I(0):=0;
    END(*IF*)
(* CALCULATE RESIDUAL SOLUTION SECTION LENGTH *)
REALDIV:=LENGTH/DIVL;
NUMDIV:=TRUNC(REALDIV);
RESL:=(REALDIV-NUMDIV)*DIVL;
IF RESL > 0 THEN
  BEGIN
    NUMDIV:=NUMDIV+1;
    END(*IF*)
ELSE
  BEGIN
    RESL:=DIVL;
    END(*ELSE*)
(* DEFINE LENGTH SECTION ARRAY *)
NUMDIV:=NUMTOT+NUMDIV;
FOR I:=NUMTOT+1 TO NUMDIV DO I(I):=I(I-1)+DIVL;
I(NUMDIV):=I(NUMDIV-1)+RESL;
I(NUMDIV+1):=I(NUMDIV)+DIVL;
END(*SECTION*)

```

```
FUNCTION STAGTEMP(T0,TW,NU,KF,MDOY,L:REAL):REAL;
(* CALCULATES NEW STAGNATION TEMPERATURE FOR FLOW WITH HEAT TRANSFER *)
BEGIN
  STAGTEMP:=TW+((T0-TW)/EXP((NU*PI*KF*L)/(MDOY*CP)));
END;(*STAGTEMP*)
```

```
FUNCTION RFUNC(M,T0,TW,PR,NU:REAL; KEY:INTEGER):REAL;
(* SOLVES AN OFTEN USED FUNCTION IN PROCEDURE HEATTRANS *)
VAR A,C,D,E:REAL;
BEGIN
  A:=M*B(M)/(2.0*(1.0-SQR(M)));
  C:=(1.0+K*SQR(M))/T0;
  D:=K*SQR(M);
  IF KEY = 1 THEN
    BEGIN
      E:=2.0/(TW-T0);
    END(*IF*)
  ELSE
    BEGIN
      E:=16.0*PR/(NU*(TW-T0));
    END(*ELSE*)
  RFUNC:=A*(D+E+C);
END;(* RFUNC *)
```

```
PROCEDURE HEATTRANS(TW,L,D,MDOY:REAL; M:INTEGER; VAR M,T0,P0,T,P,RHO,RE,F,Q,QX,NU:REAL);
(* CALCULATES FLOW VARIABLES FOR FLOW WITH FRICTION AND HEAT TRANSFER *)
```

```
VAR
  C1,C2,C3,C4,STEP,A,C:REAL;
  T01,T02,M1,M2,PR,KF,DELM,X:REAL;
  KEY,I:INTEGER;
BEGIN
  KEY:=0;
  (* FLAG KEY IF FLOW TURBULENT *)
  IF NU > 3.46 THEN KEY:=1;
  (* CALCULATE NEW STAGNATION TEMPERATURE *)
  PR:=PRANDTL(T);
  KF:=COND(T);
  T01:=T0;
  T02:=STAGTEMP(T01,TW,NU,KF,MDOY,L);
  (* STEPWISE SOLUTION USING RUNGE-KUTTE METHOD *)
  (* CALCULATE STEP SIZE *)
  STEP:=ABS((T02-T01)/M);
  M1:=M;
  I:=1;
  WHILE I <= N DO
    BEGIN
      C1:=RFUNC(M,T0,TW,PR,NU,KEY);
      IF C1 < 0 THEN C1:=0;(*FUNCTION UNSTABLE*)
      C2:=RFUNC((M+C1*STEP/2.0),(T0+STEP/2.0),TW,PR,NU,KEY);
      IF C2 < 0 THEN C2:=0;(*FUNCTION UNSTABLE*)
      C3:=RFUNC((M+C2*STEP/2.0),(T0+STEP/2.0),TW,PR,NU,KEY);
      IF C3 < 0 THEN C3:=0;(*FUNCTION UNSTABLE*)
      C4:=RFUNC(M+C3*STEP,T0+STEP,TW,PR,NU,KEY);
      IF C4 < 0 THEN C4:=0;(*FUNCTION UNSTABLE*)
      DELM:=STEP/6.0*(C1+2.0*C2+2.0*C3+C4);
      M:=M+DELM;
      T0:=T0+STEP;
      I:=I+1;
      IF M > 1 THEN (*FUNCTION UNSTABLE, EXIT LOOP*)
        BEGIN
          I:=M+1;
          M:=1.0;
        END(*IF*)
      END;(* WHILE *)
      IF (C1 = 0) OR (C2 = 0) OR (C3 = 0) OR (C4 = 0) THEN
        (*FUNCTION UNSTABLE, EXIT LOOP*)
        BEGIN
          I:=M+1;
          M:=1;
        END;(*IF*)
      M2:=M;
      A:=B(M2)/B(M1);
      C:=(K+1.0)/(2.0*(K-1.0));
      P0:=P0*(M1/M2)*SORT(T02/T01)*PWR(A,C);
```



```

P:=FNES(P0,M1);
T:=TEMP(T01,M1);
RHO:=DENSE(P,T);
RE:=4.0*MDOT/(PI*0*NU(T));
IF RE (= 1000 THEN
  BEGIN (* LAMINAR FLOW *)
    F:=16.0/RE;
    NU:=3.66;
    END(*IF*)
ELSE
  BEGIN (* TURBULENT FLOW *)
    F:=0.046/PWR(RE,0.2);
    NU:=0.023*PWR(RE,0.8)*PWR(PR,0.4);
    END(*ELSE*)
Q:=CP*MDOT*(T01-T01);
QX:=QX+Q;
END;(* HEATTRANS *)

```

```

PROCEDURE SETBOUND(KEY:INTEGER; VAR MIN,LOW,DIA:REAL);
(* ALTERS DIAMETER SO THAT MACH MAY BE INTERATED IN THE MAIN PROGRAM TO REACH ONE *)
BEGIN(*SETBOUND*)
IF KEY = 1 THEN
  BEGIN
    MIN:=DIA;
    END(*IF*)
ELSE
  BEGIN
    LOW:=DIA;
    END(*ELSE*)
DIA:=(LOW+MIN)/2.0;
END;(*SETBOUND*)

```

```

PROCEDURE SWAP(VAR A,B:REAL);
(* USED BY PROCEDURE BSORT *)
VAR T:REAL;
BEGIN(*SWAP*)
T:=A;
A:=B;
B:=T;
END;(*SWAP*)

```

```

PROCEDURE BSORT(START, TOP:INTEGER; VAR ARRY:COUNT; VAR CNT:ICOUNT);
(* USED IN FUNCTION MDOTMIN TO FIND OPTIMUM MASS FLOW *)

```

```

VAR
  RCNT:COUNT;
  INTEGER;
  SWITCH:BOOLEAN;
BEGIN(*BSORT*)
FOR I:=START TO TOP DO RCNT[I]:=CNT[I];
REPEAT(* UNTIL SORTED WITH SMALLEST AT TOP *)
  SWITCH:=FALSE;
  FOR I:=START TO TOP-1 DO
    BEGIN
      IF ARRY[I] > ARRY[I+1] THEN
        BEGIN
          SWAP(ARRY[I],ARRY[I+1]);
          SWAP(RCNT[I],RCNT[I+1]);
          SWITCH:=TRUE;
        END(*IF*)
      END;(*FOR*)
  UNTIL SWITCH = FALSE;
  FOR I:=START TO TOP DO CNT[I]:=ROUND(RCNT[I]);
END;(*BSORT*)

```

```

FUNCTION MDOTMIN(Q,TW:COUNT; T01,EFF:REAL; N:INTEGER):REAL;
(* DETERMINES OPTIMUM MASS FLOW *)

```

```

VAR
  QTOT,TOMAX,MDOT:COUNT;
  INDEX:ICOUNT;
  I,NCNT:INTEGER;

```

```

BEGIN(*MDOOTMIN*)
(* INITIALIZE INDEX ARRAY *)
FOR I:=0 TO N DO INDEX(I):=1;
(* CALCULATE SHIELD EXIT STAGNATION TEMPERATURES *)
TMAX(0):=T01;
FOR I:=1 TO N DO TMAX(I):=EFF*(TV(I)-TMAX(I-1))+TMAX(I-1);
(* CALCULATE MASS FLOWS AS DETERMINED BY SECTIONS *)
QTOT(0):=0;
FOR I:=1 TO N DO
  BEGIN
    QTOT(I):=QTOT(I-1)+Q(I);
    NTWT(I):=QTOT(I)/(CP*(TMAX(I)-T01));
  END;(*FOR*)
(* SEARCH FOR LARGEST MDOOT *)
BSORT(1,N,MDOOT,INDEX);
MDOOTMIN:=MDOOT(N);
END;(*MDOOTMIN*)

```

```

PROCEDURE INREASSIGN;
(*DEFINES VALUES FOR USE IN PARAMETER LIST *)
BEGIN(*INREASSIGN*)
  M1:=M(1);
  T01:=T(0);
  P01:=P(0);
  T1:=T(1);
  P1:=P(1);
  RHO1:=RHO(1);
  F1:=F(1);
  RE1:=RE(1);
  Q1:=Q(1);
  QX1:=QX(1);
  NU1:=NU(1);
END;(*INREASSIGN*)

```

```

PROCEDURE OUTREASSIGN;
(* DEFINES NEW VALUES OBTAINED FROM PARAMETER LIST *)
BEGIN(*OUTREASSIGN*)
  I:=I+1;
  M(I):=M1;
  T(1):=T01;
  P(1):=P01;
  T(I):=T1;
  P(I):=P1;
  RHO(I):=RHO1;
  F(I):=F1;
  RE(I):=RE1;
  Q(I):=Q1;
  QX(I):=QX1;
  NU(I):=NU1;
END;(*OUTREASSIGN*)

```

```

PROCEDURE INITIALIZE;
(* PREPARES VARIABLES FOR USE IN MAIN PROGRAM *)
BEGIN(*INITIALIZE*)
(*DETERMINE MINIMUM MASS FLOW*)
MDOOT:=MDOOTMIN(QV,TV,T(1),MAXEFF,NUMWALLS);
(* DEFINE FIRST SECTION SIZE *)
X(1):=DIVLENGTH;
(* DEFINE IMAGINARY LAST PIPE LENGTH *)
LSPACE(NUMWALLS+2):=0;
(*START LOOP COUNTER*)
LOOPCOUNT:=0;
(*INITIAL MACH NUMBER GUESS*)
M(1):=0.1;
(* INITIAL DIAMETER GUESS, ASSUMES INLET MACH OF 0.1 *)
DIAM(1):=DIAMETER(MDOOT,T(1),P(1),M(1));
(* CALCULATE MINIMUM DIAMETER, MACH = 1 *)
LOWBOUND:=DIAMETER(MDOOT,T(1),P(1),1.0);
(* DEFINE MAXIMUM DIAMETER *)
HIBOUND:=100*LOWBOUND;
END;(*INITIALIZE*)

```

```

PROCEDURE REINITIAL;
(* PREPARES VARIABLES FOR USE IN MAIN PROGRAM *)
BEGIN(*REINITIAL*)

```

```
(*INITIALIZE COUNTERS*)
I:=1;
PIPECNT:=1;
WALLCNT:=1;
SPACECNT:=1;
(*INCREMENT LOOP COUNTER*)
LOOPCOUNT:=LOOPCOUNT+1;
(* DEFINE FIRST PIPE LENGTH *)
LTOTAL:=LSPACE(1);
(*CALCULATE ENTRANCE CONDITIONS*)
QX(1):=0;Q(1):=0;
RE(1):=4.0*MDOT/(PI*NU(T0(1))*DIA(LOOPCOUNT));
IF RE(1) <= 2000 THEN
  BEGIN (* LAMINAR FLOW *)
    F(1):=16.0/RE(1);
    NU(1):=3.44;
  END(*IF*)
ELSE
  BEGIN (* TURBULENT FLOW *)
    F(1):=0.046/PWR(RE(1),0.2);
    NU(1):=0.023*PWR(RE(1),0.8)*PWR(PRANDTL(T0(1)),0.4);
  END(*ELSE*)
(*CALCULATE INLET CONDITIONS*)
INLETMACH(DIA(LOOPCOUNT),MDOT,P0(1),T0(1),NU(1),T(1),P(1),RHO(1));
(*INITIALIZE CONTROL CARD*)
MTEST:=1;
END;(*REINITIAL*)
```

```
BEGIN(* MAIN PROGRAM *)
  WRITELN('TO ENTER DATA, ENTER 1. OTHERWISE, ENTER 0. ');
  READLN;
  READ(TAB);
  INDATA(TAB);
  INITIALIZE;

  REPEAT (* UNTIL THE FLOW IS CHOKED *)
    REINITIAL;
    WHILE (SPACECNT (= NUMWALLS+1) AND (MTEST < 2) DO
      BEGIN
        (* TEST FOR CHANGE IN TEMPERATURE BOUNDARY *)
        IF X(1) < (LTOTAL-DIVLENGTH/2.0) THEN
          BEGIN(* FRICTION FLOW *)
            (* DIVIDE FRICTION LENGTH INTO SECTIONS *)
            SECTION(1,LSPACE(SPACECNT),DIVLENGTH,X,NUMDIV);
            (* DEFINE END MACH NUMBER FOR TEST AT END *)
            MINUMDIV:=0;
            WHILE (1 < NUMDIV) AND (MTEST < 2) DO
              BEGIN
                L:=X(1)-X(1-1);
                INREASSIGN;
                FRICTION(T01,DIA(LOOPCOUNT),F1,L,M1,T1,P1,RHO1,P01,Q1);
                OUTREASSIGN;
                IF M(1) > 0.97 THEN MTEST:=2 (*EXIT LOOP*)
                END;(*WHILE*)
              SPACECNT:=SPACECNT+1;
              PIPECNT:=PIPECNT+1;
              END(* FRICTION FLOW *)
            ELSE
              BEGIN(* TEMPERATURE BOUNDARY CHANGE *)
                QWALL(WALLCNT):=0;
                LTEMP(WALLCNT):=0;
                EFF(WALLCNT):=0;
                TOSAVE:=T0(1);
                WHILE (QW(WALLCNT) > QWALL(WALLCNT)) AND (EFF(WALLCNT) < MAXEFF) AND (MTEST < 2) DO
                  BEGIN
                    INREASSIGN;
                    HEATTRANS(TW(WALLCNT),DIVLENGTH,DIA(LOOPCOUNT),MDOT,N,M1,T01,P01,T1,P1,RHO1,RE1,F1,Q1,QX1,NU1);
                    OUTREASSIGN;
                    QWALL(WALLCNT):=QW(WALLCNT)+Q1;
                    EFF(WALLCNT):=(T0(1)-TOSAVE)/(TW(WALLCNT)-TOSAVE);
                    LTEMP(WALLCNT):=LTEMP(WALLCNT)+DIVLENGTH;
                    X(1):=X(1-1)+DIVLENGTH;
                    IF M(1) > 0.97 THEN MTEST:=3;(*EXIT LOOPS*)
                  END;(*WHILE*)
                LTOTAL:=LTOTAL+LTEMP(WALLCNT)+LSPACE(SPACECNT);
                PIPECNT:=PIPECNT+1;
                WALLCNT:=WALLCNT+1;
                END;(* TEMPERATURE BOUNDARY CHANGE *)
              END;(*WHILE*)
          END;(*WHILE*)
        END;(*WHILE*)
      END;(*WHILE*)
    END;(*WHILE*)
  END;(*WHILE*)
END;(*WHILE*)
```

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```
DIA1:=DIALOOPCOUNT;  
SETHOUND(MTEST,HIBOUND,LOWBOUND,DIA1);  
DIALOOPCOUNT+1:=DIA1;  
UNTIL (MINUMDIV) > 0.97) AND (PIPECNT)= 2*NUMVALLS+1) OR (LOOPCOUNT > 30);
```

```
(*CALCULATE OVERALL EFFECTIVENESS *)  
EFF(NUMVALLS+1):=(T0(NUMDIV)-T0(1))/(TV(NUMVALLS)-T0(1));  
(* CALCULATE TOTAL HEAT TRANSFER *)  
QWALL(NUMVALLS+1):=MDOT*CP*(T0(NUMDIV)-T0(1));  
WRITELN('OUTPUT STORED IN FILE EFFOUTD.');
```

```
WRITELN('INPUT STORED IN FILES EFFIND1 AND EFFIND2.');
```

```
OUTDATA;
```

```
END. (*MAIN PROGRAM*)
```

BIND.
TO ENTER DATA, ENTER 1. OTHERWISE, ENTER 0.
? 1
INPUT DATA AS PROMPTED BY TERMINAL

WHEN ASKED FOR A SERIES OF DATA SUCH AS THE SHIELD
TEMPERATURES, INPUT THE DATA IN THE ORDER FROM DEWAR
TO THE OUTER BOUNDARY.

ENTER THE NUMBER OF RADIATION SHIELDS
? 5
NUMBER OF SHIELDS= 5

ENTER SINGLE DIVISION LENGTH (M)
? 0.002
DIVISION LENGTH (M)= 0.00200

ENTER PIPE LENGTHS NOT ON SHIELDS (M)
? 0.019
? 0.021
? 0.024
? 0.026
? 0.034
? 0.035

ENTER TEMPERATURES OF RADIATION SHIELDS (K)
? 21.7
? 39.7
? 65.4
? 101.1
? 146.6

ENTER HEAT INPUTS OF RADIATION SHIELDS (W)
? 0.0884
? 0.054
? 0.0916
? 0.1386
? 0.8242

ENTER MAXIMUM HEAT EXCHANGER EFFECTIVENESS AS A FRACTION
? 0.98
SHIELD EFFECTIVENESS= 0.9800000

ENTER INITIAL STAGNATION T (K)
? 4.23
INITIAL STAGNATION TEMPERATURE (K)= 4.23600

ENTER INITIAL STAGNATION P (PA)
? 2100.0
INITIAL STAGNATION PRESSURE (PA)= 2100.00000

ENTER NUMBER OF STEPS IN RUNGE-KUTTA METHOD
? 4

STEPS IN RUNGE-KUTTA= 4
OUTPUT STORED IN FILE EFFOUTD.
INPUT STORED IN FILES EFFIND1 AND EFFIND2.
FOR AN OUTPUT OF SECTION FLOW VARIABLES, INPUT 1. OTHERWISE INPUT 0.
? 1
FOR A PLOT FILE ENTER 1, OTHERWISE ENTER 0
? 0

NUMBER OF INTERACTIONS= 13
FOR INTERACTION INFORMATION, INPUT 1. OTHERWISE INPUT 0
? 1
2.806 CP SECS, 1027168 CM USED.

VALUES OF INPUT

INLET STAGNATION TEMPERATURE (K)= 4.2300000000
 INLET STAGNATION PRESSURE (PA)=2100.0000000000
 NUMBER OF RADIATION SHIELDS= 5

TEMPERATURES OF RADIATION SHIELDS
 SHIELD TEMPERATURE (K) 1= 21.7000000000
 SHIELD TEMPERATURE (K) 2= 39.7000000000
 SHIELD TEMPERATURE (K) 3= 65.4000000000
 SHIELD TEMPERATURE (K) 4= 101.1000000000
 SHIELD TEMPERATURE (K) 5= 146.4000000000

HEAT INPUTS TO RADIATION SHIELDS
 SHIELD HEAT (W) 1= 0.0000000000
 SHIELD HEAT (W) 2= 0.0540000000
 SHIELD HEAT (W) 3= 0.0916000000
 SHIELD HEAT (W) 4= 0.1384000000
 SHIELD HEAT (W) 5= 0.2242000000

PIPE LENGTHS BETWEEN SHIELDS
 SPACE LENGTH (M) 1= 0.0190000000
 SPACE LENGTH (M) 2= 0.0210000000
 SPACE LENGTH (M) 3= 0.0240000000
 SPACE LENGTH (M) 4= 0.0260000000
 SPACE LENGTH (M) 5= 0.0340000000
 SPACE LENGTH (M) 6= 0.0350000000

SOLUTION SECTION SIZE (M)= 0.0020000000
 MAXIMUM SHIELD EXCHANGER EFFECTIVENESS= 0.9000000000

VALUES OF OUTPUT

MASS FLOW (KG/S)= 0.0000016272
 PIPE DIAMETER (M)= 0.0018244634

TOTAL PIPE LENGTH (M)= 0.2530000000
 SHIELD LENGTHS

SHIELD LENGTH (M) 1= 0.0320000000
 SHIELD LENGTH (M) 2= 0.0100000000
 SHIELD LENGTH (M) 3= 0.0080000000
 SHIELD LENGTH (M) 4= 0.0040000000
 SHIELD LENGTH (M) 5= 0.0380000000

TOTAL HEAT TRANSFER (W)= 1.1923520433

SHIELD HEAT TRANSFER

SHIELD HEAT (W) 1= 0.0904974340
 SHIELD HEAT (W) 2= 0.0442480135
 SHIELD HEAT (W) 3= 0.1081724340
 SHIELD HEAT (W) 4= 0.1577594101
 SHIELD HEAT (W) 5= 0.7714723449

OVERALL EXCHANGER EFFECTIVENESS= 0.9098140440
 SHIELD EFFECTIVENESS

SHIELD EFF 1= 0.4135927631
 SHIELD EFF 2= 0.3047910789
 SHIELD EFF 3= 0.2983848314
 SHIELD EFF 4= 0.2834763520
 SHIELD EFF 5= 0.9843448598

EXIT STAGNATION TEMPERATURE (K)= 145.1498257355
 EXIT STAGNATION PRESSURE (PA)= 449.2579023906

LENGTH	MACH NUMBER	STAG TEMP	STAG PRES	TEMP	PRESSURE	MUSSELT	REYNOLD NUM	SECTION HEAT	TOTAL HEAT
0.00200	0.02145443	4.230000	2100.00000	4.2293	2099.193	3.660	713.65327	0.0000000	0.0000000
0.00400	0.02149779	4.230000	2095.97130	4.2293	2095.163	3.660	713.65327	0.0000000	0.0000000
0.00600	0.02153925	4.230000	2091.95004	4.2293	2091.140	3.660	713.65327	0.0000000	0.0000000

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0.00000	0.02150048	4.230000	2087.93629	4.2293	2087.125	3.460	713.65327	0.0000000	0.0000000
0.01000	0.02142220	4.230000	2083.92994	4.2293	2083.117	3.460	713.65327	0.0000000	0.0000000
0.01200	0.02146377	4.230000	2079.93104	4.2293	2079.114	3.460	713.65327	0.0000000	0.0000000
0.01400	0.02170547	4.230000	2075.93957	4.2293	2075.123	3.460	713.65327	0.0000000	0.0000000
0.01600	0.02174724	4.230000	2071.93549	4.2293	2071.137	3.460	713.65327	0.0000000	0.0000000
0.01800	0.02178900	4.230000	2067.97879	4.2293	2067.159	3.460	713.65327	0.0000000	0.0000000
0.01900	0.02183101	4.230000	2064.80946	4.2293	2063.188	3.460	713.65327	0.0000000	0.0000000
0.02100	0.02393310	5.001781	2063.72487	5.0008	2062.740	3.460	672.91490	0.0072071	0.0072071
0.02300	0.02502612	5.914977	2063.42159	5.9137	2062.273	3.460	637.46097	0.0070490	0.0142549
0.02500	0.02754924	6.727647	2063.89309	6.7259	2061.786	3.460	606.44139	0.0048762	0.0211331
0.02700	0.02912741	7.518072	2062.74092	7.5159	2061.280	3.460	579.15677	0.0046879	0.0278211
0.02900	0.03058602	8.284762	2062.36478	8.2822	2060.755	3.460	555.03770	0.0044871	0.0343082
0.03100	0.03193379	9.024463	2061.94446	9.0234	2060.210	3.460	533.63239	0.0042757	0.0405039
0.03300	0.03318426	9.742153	2061.53991	9.7384	2059.646	3.460	514.54304	0.0040556	0.0464395
0.03500	0.03434474	10.431042	2061.89116	10.4269	2059.862	3.460	497.51821	0.0038288	0.0524683
0.03700	0.03542904	11.092540	2060.41836	11.0879	2058.460	3.460	482.23767	0.0035972	0.0580655
0.03900	0.03643766	11.726346	2060.12177	11.7211	2057.840	3.460	468.50226	0.0033626	0.0634281
0.04100	0.03737833	12.332237	2059.60174	12.3265	2057.281	3.460	456.12612	0.0031266	0.0685547
0.04300	0.03825605	12.910240	2059.05817	12.9039	2056.545	3.460	444.95061	0.0028908	0.0734455
0.04500	0.03907531	13.460593	2058.49318	13.4537	2055.871	3.460	434.83954	0.0026456	0.0781020
0.04700	0.03984014	13.983576	2057.90573	13.9761	2055.181	3.460	425.67537	0.0024021	0.0825270
0.04900	0.04055424	14.479673	2057.29700	14.4717	2054.474	3.460	417.35605	0.0021676	0.0867246
0.05100	0.04122099	14.949466	2056.66767	14.9410	2053.753	3.460	409.79270	0.0019375	0.0906996
0.05300	0.04172748	14.949466	2055.99041	14.9409	2053.071	3.460	409.79270	0.0017070	0.0940996
0.05500	0.04213287	14.949466	2055.31545	14.9409	2052.393	3.460	409.79270	0.0014765	0.0969996
0.05700	0.04253828	14.949466	2054.64279	14.9409	2051.716	3.460	409.79270	0.0012460	0.0994996
0.05900	0.04294369	14.949466	2053.97241	14.9409	2051.042	3.460	409.79270	0.0010155	0.1019996
0.06100	0.04334911	14.949466	2053.30433	14.9408	2050.370	3.460	409.79270	0.0007850	0.1044996
0.06300	0.04375452	14.949466	2052.63853	14.9408	2049.700	3.460	409.79270	0.0005545	0.1069996
0.06500	0.04415994	14.949466	2051.97501	14.9408	2049.033	3.460	409.79270	0.0003240	0.1094996
0.06700	0.04456536	14.949466	2051.31377	14.9408	2048.366	3.460	409.79270	0.0000935	0.1119996
0.06900	0.04497078	14.949466	2050.65480	14.9408	2047.705	3.460	409.79270	0.0000000	0.1144996
0.07100	0.04537620	14.949466	2049.99810	14.9407	2047.045	3.460	409.79270	0.0000000	0.1169996
0.07200	0.04588162	14.949466	2049.34367	14.9407	2046.386	3.460	409.79270	0.0000000	0.1194996
0.07400	0.04638704	16.577946	2048.68924	16.5672	2045.726	3.460	385.71186	0.0137791	0.1219996
0.07600	0.04689246	18.157728	2048.03481	18.1448	2045.067	3.460	365.08728	0.0133666	0.1244996
0.07800	0.04739788	19.681992	2047.38038	19.6668	2044.408	3.460	347.32619	0.0129541	0.1269996
0.08000	0.04790330	21.145152	2046.72595	21.1276	2043.749	3.460	331.95665	0.0125416	0.1294996
0.08200	0.04840872	22.542709	2046.07152	22.5227	2043.090	3.460	318.68058	0.0121291	0.1319996
0.08400	0.04891414	22.542709	2045.41709	22.5226	2042.431	3.460	318.68058	0.0117166	0.1344996
0.08600	0.04941956	22.542709	2044.76266	22.5226	2041.772	3.460	318.68058	0.0113041	0.1369996
0.08800	0.04992498	22.542709	2044.10823	22.5225	2041.113	3.460	318.68058	0.0108916	0.1394996
0.09000	0.05043040	22.542709	2043.45380	22.5224	2040.454	3.460	318.68058	0.0104791	0.1419996
0.09200	0.05093582	22.542709	2042.79937	22.5224	2039.795	3.460	318.68058	0.0100666	0.1444996
0.09400	0.05144124	22.542709	2042.14494	22.5223	2039.136	3.460	318.68058	0.0096541	0.1469996
0.09600	0.05194666	22.542709	2041.49051	22.5222	2038.477	3.460	318.68058	0.0092416	0.1494996
0.09800	0.05245208	22.542709	2040.83608	22.5222	2037.818	3.460	318.68058	0.0088291	0.1519996
0.10000	0.05295750	22.542709	2040.18165	22.5222	2037.159	3.460	318.68058	0.0084166	0.1544996
0.10200	0.05346292	22.542709	2039.52722	22.5222	2036.500	3.460	318.68058	0.0080041	0.1569996
0.10400	0.05396834	22.542709	2038.87279	22.5221	2035.841	3.460	318.68058	0.0075916	0.1594996
0.10600	0.05447376	22.542709	2038.21836	22.5221	2035.182	3.460	318.68058	0.0071791	0.1619996
0.10800	0.05497918	22.542709	2037.56393	22.5221	2034.523	3.460	318.68058	0.0067666	0.1644996
0.11000	0.05548460	22.542709	2036.90950	22.5221	2033.864	3.460	318.68058	0.0063541	0.1669996
0.11200	0.05599002	22.542709	2036.25507	22.5221	2033.205	3.460	318.68058	0.0059416	0.1694996
0.11400	0.05649544	22.542709	2035.60064	22.5221	2032.546	3.460	318.68058	0.0055291	0.1719996
0.11600	0.05700086	22.542709	2034.94621	22.5221	2031.887	3.460	318.68058	0.0051166	0.1744996
0.11800	0.05750628	22.542709	2034.29178	22.5221	2031.228	3.460	318.68058	0.0047041	0.1769996
0.12000	0.05801170	22.542709	2033.63735	22.5221	2030.569	3.460	318.68058	0.0042916	0.1794996
0.12200	0.05851712	22.542709	2032.98292	22.5221	2029.910	3.460	318.68058	0.0038791	0.1819996
0.12400	0.05902254	22.542709	2032.32849	22.5221	2029.251	3.460	318.68058	0.0034666	0.1844996
0.12600	0.05952796	22.542709	2031.67406	22.5221	2028.592	3.460	318.68058	0.0030541	0.1869996
0.12800	0.06003338	22.542709	2031.01963	22.5221	2027.933	3.460	318.68058	0.0026416	0.1894996
0.13000	0.06053880	22.542709	2030.36520	22.5221	2027.274	3.460	318.68058	0.0022291	0.1919996
0.13200	0.06104422	22.542709	2029.71077	22.5221	2026.615	3.460	318.68058	0.0018166	0.1944996
0.13400	0.06154964	22.542709	2029.05634	22.5221	2025.956	3.460	318.68058	0.0014041	0.1969996
0.13600	0.06205506	22.542709	2028.40191	22.5221	2025.297	3.460	318.68058	0.0009916	0.1994996
0.13800	0.06256048	22.542709	2027.74748	22.5221	2024.638	3.460	318.68058	0.0005791	0.2019996
0.14000	0.06306590	22.542709	2027.09305	22.5221	2023.979	3.460	318.68058	0.0001666	0.2044996
0.14200	0.06357132	22.542709	2026.43862	22.5221	2023.320	3.460	318.68058	0.0000000	0.2069996
0.14400	0.06407674	22.542709	2025.78419	22.5221	2022.661	3.460	318.68058	0.0000000	0.2094996
0.14600	0.06458216	22.542709	2025.12976	22.5221	2022.002	3.460	318.68058	0.0000000	0.2119996
0.14800	0.06508758	22.542709	2024.47533	22.5221	2021.343	3.460	318.68058	0.0000000	0.2144996
0.15000	0.06559300	22.542709	2023.82090	22.5221	2020.684	3.460	318.68058	0.0000000	0.2169996
0.15200	0.06609842	22.542709	2023.16647	22.5221	2020.025	3.460	318.68058	0.0000000	0.2194996
0.15400	0.06660384	22.542709	2022.51204	22.5221	2019.366	3.460	318.68058	0.0000000	0.2219996
0.15600	0.06710926	22.542709	2021.85761	22.5221	2018.707	3.460	318.68058	0.0000000	0.2244996
0.15800	0.06761468	22.542709	2021.20318	22.5221	2018.048	3.460	318.68058	0.0000000	0.2269996
0.16000	0.06812010	22.542709	2020.54875	22.5221	2017.389	3.460	318.68058	0.0000000	0.2294996
0.16200	0.06862552	22.542709	2019.89432	22.5221	2016.730	3.460	318.68058	0.0000000	0.2319996
0.16400	0.06913094	22.542709	2019.23989	22.5221	2016.071	3.460	318.68058	0.0000000	0.2344996
0.16600	0.06963636	22.542709	2018.58546	22.5221	2015.412	3.460	318.68058	0.0000000	0.2369996
0.16800	0.07014178	22.542709	2017.93103	22.5221	2014.753	3.460	318.68058	0.0000000	0.2394996
0.17000	0.07064720	22.542709	2017.27660	22.5221	2014.094	3.460	318.68058	0.0000000	0.2419996
0.17200	0.07115262	22.542709	2016.62217	22.5221	2013.435	3.460	318.68058	0.0000000	0.2444996
0.17400	0.07165804	22.542709	2015.96774	22.5221	2012.776	3.460	318.68058	0.0000000	0.2469996
0.17600	0.07216346	22.542709	2015.31331	22.5221	2012.117	3.460	318.68058	0.0000000	0.2494996
0.17800	0.07266888	22.542709	2014.65888	22.5221	2011.458	3.460	318.68058	0.0000000	0.2519996
0.18000	0.07317430	22.542709	2014.00445	22.5221	2010.799	3.460	318.68058	0.0000000	0.2544996
0.18200	0.07367972	22.542709	2013.35002	22.5221	2010.140	3.460	318.68058	0.0000000	0.2569996
0.18400	0.07418514	22.542709	2012.69						

0.16400	0.08722128	53.972266	1854.14068	53.8351	1842.415	3.660	174.69468	0.0000000	0.4208797
0.16800	0.08748881	53.972266	1848.52835	53.8342	1836.766	3.660	174.69468	0.0000000	0.4208797
0.17200	0.08775850	53.972266	1842.90572	53.8336	1831.108	3.660	174.69468	0.0000000	0.4208797
0.17600	0.08803037	53.972266	1837.27266	53.8325	1825.430	3.660	174.69468	0.0000000	0.4208797
0.18000	0.08830444	53.972266	1831.62908	53.8316	1819.757	3.660	174.69468	0.0000000	0.4208797
0.18400	0.08858077	53.972266	1825.97487	53.8308	1814.066	3.660	174.69468	0.0000000	0.4208797
0.18800	0.08885936	53.972266	1820.30992	53.8299	1808.366	3.660	174.69468	0.0000000	0.4208797
0.19200	0.08914026	53.972266	1814.63411	53.8290	1802.650	3.660	174.69468	0.0000000	0.4208797
0.19600	0.08942408	53.972266	1808.94984	53.8280	1796.922	3.660	174.69468	0.0000000	0.4208797
0.20000	0.08971037	53.972266	1803.25610	53.8271	1791.194	3.660	174.69468	0.0000000	0.4208797
0.20400	0.09000000	53.972266	1797.56237	53.8262	1785.466	3.660	174.69468	0.0000000	0.4208797
0.20800	0.09029233	53.972266	1791.86864	53.8253	1779.738	3.660	174.69468	0.0000000	0.4208797
0.21200	0.09058722	53.972266	1786.17491	53.8244	1774.010	3.660	174.69468	0.0000000	0.4208797
0.21600	0.09088466	53.972266	1780.48118	53.8235	1768.282	3.660	174.69468	0.0000000	0.4208797
0.22000	0.09118466	53.972266	1774.78745	53.8226	1762.554	3.660	174.69468	0.0000000	0.4208797
0.22400	0.09148722	53.972266	1769.09372	53.8217	1756.826	3.660	174.69468	0.0000000	0.4208797
0.22800	0.09179233	53.972266	1763.39999	53.8208	1751.098	3.660	174.69468	0.0000000	0.4208797
0.23200	0.09210000	53.972266	1757.70626	53.8199	1745.370	3.660	174.69468	0.0000000	0.4208797
0.23600	0.09241022	53.972266	1752.01253	53.8190	1739.642	3.660	174.69468	0.0000000	0.4208797
0.24000	0.09272300	53.972266	1746.31880	53.8181	1733.914	3.660	174.69468	0.0000000	0.4208797
0.24400	0.09303833	53.972266	1740.62507	53.8172	1728.186	3.660	174.69468	0.0000000	0.4208797
0.24800	0.09335622	53.972266	1734.93134	53.8163	1722.458	3.660	174.69468	0.0000000	0.4208797
0.25200	0.09367666	53.972266	1729.23761	53.8154	1716.730	3.660	174.69468	0.0000000	0.4208797
0.25600	0.09399966	53.972266	1723.54388	53.8145	1711.002	3.660	174.69468	0.0000000	0.4208797
0.26000	0.09432522	53.972266	1717.85015	53.8136	1705.274	3.660	174.69468	0.0000000	0.4208797
0.26400	0.09465333	53.972266	1712.15642	53.8127	1699.546	3.660	174.69468	0.0000000	0.4208797
0.26800	0.09498400	53.972266	1706.46269	53.8118	1693.818	3.660	174.69468	0.0000000	0.4208797
0.27200	0.09531722	53.972266	1700.76896	53.8109	1688.090	3.660	174.69468	0.0000000	0.4208797
0.27600	0.09565400	53.972266	1695.07523	53.8100	1682.362	3.660	174.69468	0.0000000	0.4208797
0.28000	0.09599422	53.972266	1689.38150	53.8091	1676.634	3.660	174.69468	0.0000000	0.4208797
0.28400	0.09633800	53.972266	1683.68777	53.8082	1670.906	3.660	174.69468	0.0000000	0.4208797
0.28800	0.09668522	53.972266	1677.99404	53.8073	1665.178	3.660	174.69468	0.0000000	0.4208797
0.29200	0.09703600	53.972266	1672.30031	53.8064	1659.450	3.660	174.69468	0.0000000	0.4208797
0.29600	0.09739022	53.972266	1666.60658	53.8055	1653.722	3.660	174.69468	0.0000000	0.4208797
0.30000	0.09774800	53.972266	1660.91285	53.8046	1647.994	3.660	174.69468	0.0000000	0.4208797
0.30400	0.09810922	53.972266	1655.21912	53.8037	1642.266	3.660	174.69468	0.0000000	0.4208797
0.30800	0.09847400	53.972266	1649.52539	53.8028	1636.538	3.660	174.69468	0.0000000	0.4208797
0.31200	0.09884222	53.972266	1643.83166	53.8019	1630.810	3.660	174.69468	0.0000000	0.4208797
0.31600	0.09921400	53.972266	1638.13793	53.8010	1625.082	3.660	174.69468	0.0000000	0.4208797
0.32000	0.09958922	53.972266	1632.44420	53.8001	1619.354	3.660	174.69468	0.0000000	0.4208797
0.32400	0.09996800	53.972266	1626.75047	53.7992	1613.626	3.660	174.69468	0.0000000	0.4208797
0.32800	0.10035022	53.972266	1621.05674	53.7983	1607.898	3.660	174.69468	0.0000000	0.4208797
0.33200	0.10073600	53.972266	1615.36301	53.7974	1602.170	3.660	174.69468	0.0000000	0.4208797
0.33600	0.10112522	53.972266	1609.66928	53.7965	1596.442	3.660	174.69468	0.0000000	0.4208797
0.34000	0.10151800	53.972266	1603.97555	53.7956	1590.714	3.660	174.69468	0.0000000	0.4208797
0.34400	0.10191422	53.972266	1598.28182	53.7947	1584.986	3.660	174.69468	0.0000000	0.4208797
0.34800	0.10231400	53.972266	1592.58809	53.7938	1579.258	3.660	174.69468	0.0000000	0.4208797
0.35200	0.10271722	53.972266	1586.89436	53.7929	1573.530	3.660	174.69468	0.0000000	0.4208797
0.35600	0.10312400	53.972266	1581.20063	53.7920	1567.802	3.660	174.69468	0.0000000	0.4208797
0.36000	0.10353422	53.972266	1575.50690	53.7911	1562.074	3.660	174.69468	0.0000000	0.4208797
0.36400	0.10394800	53.972266	1569.81317	53.7902	1556.346	3.660	174.69468	0.0000000	0.4208797
0.36800	0.10436522	53.972266	1564.11944	53.7893	1550.618	3.660	174.69468	0.0000000	0.4208797
0.37200	0.10478600	53.972266	1558.42571	53.7884	1544.890	3.660	174.69468	0.0000000	0.4208797
0.37600	0.10521022	53.972266	1552.73198	53.7875	1539.162	3.660	174.69468	0.0000000	0.4208797
0.38000	0.10563800	53.972266	1547.03825	53.7866	1533.434	3.660	174.69468	0.0000000	0.4208797
0.38400	0.10606922	53.972266	1541.34452	53.7857	1527.706	3.660	174.69468	0.0000000	0.4208797
0.38800	0.10650400	53.972266	1535.65079	53.7848	1521.978	3.660	174.69468	0.0000000	0.4208797
0.39200	0.10694222	53.972266	1529.95706	53.7839	1516.250	3.660	174.69468	0.0000000	0.4208797
0.39600	0.10738400	53.972266	1524.26333	53.7830	1510.522	3.660	174.69468	0.0000000	0.4208797
0.40000	0.10782922	53.972266	1518.56960	53.7821	1504.794	3.660	174.69468	0.0000000	0.4208797
0.40400	0.10827800	53.972266	1512.87587	53.7812	1499.066	3.660	174.69468	0.0000000	0.4208797
0.40800	0.10873022	53.972266	1507.18214	53.7803	1493.338	3.660	174.69468	0.0000000	0.4208797
0.41200	0.10918600	53.972266	1501.48841	53.7794	1487.610	3.660	174.69468	0.0000000	0.4208797
0.41600	0.10964522	53.972266	1495.79468	53.7785	1481.882	3.660	174.69468	0.0000000	0.4208797
0.42000	0.11010800	53.972266	1490.10095	53.7776	1476.154	3.660	174.69468	0.0000000	0.4208797
0.42400	0.11057422	53.972266	1484.40722	53.7767	1470.426	3.660	174.69468	0.0000000	0.4208797
0.42800	0.11104400	53.972266	1478.71349	53.7758	1464.698	3.660	174.69468	0.0000000	0.4208797
0.43200	0.11151822	53.972266	1473.01976	53.7749	1458.970	3.660	174.69468	0.0000000	0.4208797
0.43600	0.11199600	53.972266	1467.32603	53.7740	1453.242	3.660	174.69468	0.0000000	0.4208797
0.44000	0.11247822	53.972266	1461.63230	53.7731	1447.514	3.660	174.69468	0.0000000	0.4208797
0.44400	0.11296400	53.972266	1455.93857	53.7722	1441.786	3.660	174.69468	0.0000000	0.4208797
0.44800	0.11345422	53.972266	1450.24484	53.7713	1436.058	3.660	174.69468	0.0000000	0.4208797
0.45200	0.11394800	53.972266	1444.55111	53.7704	1430.330	3.660	174.69468	0.0000000	0.4208797
0.45600	0.11444622	53.972266	1438.85738	53.7695	1424.602	3.660	174.69468	0.0000000	0.4208797
0.46000	0.11494800	53.972266	1433.16365	53.7686	1418.874	3.660	174.69468	0.0000000	0.4208797
0.46400	0.11545422	53.972266	1427.46992	53.7677	1413.146	3.660	174.69468	0.0000000	0.4208797
0.46800	0.11596400	53.972266	1421.77619	53.7668	1407.418	3.660	174.69468	0.0000000	0.4208797
0.47200	0.11647822	53.972266	1416.08246	53.7659	1401.690	3.660	174.69468	0.0000000	0.4208797
0.47600	0.11699600	53.972266	1410.38873	53.7650	1395.962	3.660	174.69468	0.0000000	0.4208797
0.48000	0.11751822	53.972266	1404.69500	53.7641	1390.234	3.660	174.69468	0.0000000	0.4208797
0.48400	0.11804400	53.972266	1399.00127	53.7632	1384.506	3.660	174.69468	0.0000000	0.4208797
0.48800	0.11857422	53.972266	1393.30754	53.7623	1378.778	3.660	174.69468	0.0000000	0.4208797
0.49200	0.11910800	53.972266	1387.61381	53.7614	1373.050	3.660	174.69468	0.0000000	0.4208797
0.49600	0.11964622	53.972266	1381.92008	53.7605	1367.322	3.660	174.69468	0.0000000	0.4208797
0.50000	0.12018900	53.972266	1376.22635	53.7596	1361.594	3.660	174.69468	0.0000000	0.4208797
0.50400	0.12073622	53.972266	1370.53262	53.7587	1355.866	3.660	174.69468	0.0000000	0.4208797
0.50800	0.12128800	53.972266	1364.83889	53.7578	1350.138	3.660	174.69468	0.0000000	0.4208797
0.51200	0.12184422	53.972266	1359.14516	53.7569	1344.410	3.660	174.69468	0.0000000	0.4208797
0.51600	0.12240500	53.972266	1353.45143	53.7560	1338.682	3.660	174.69468	0.0000000	0.4208797
0.52000	0.12297022	53.972266	1347.75770	53.7551					

27

```

TYPE, EFFLUXM
PROGRAM EFFLUXM(INPUT/,OUTPUT,EFFOUTH,EFFINM1,EFFINM2,POUT);

```

ORIGINAL PAGE IS
OF POOR QUALITY

[illegible]

```

(.....)
(.....)
(..... J. C. CHATO AND K. W. BRENDLEY ..... )
(.....)
(.....)
(..... UNIVERSITY OF ILLINOIS ..... )
(.....)
(..... UNDER CONTRACT OF NASA, AMES RESEARCH ..... )
(.....)
(..... JULY 1, 1962 ..... )
(.....)
(.....)
(.....)

```

(*) THIS PROGRAM DETERMINES CERTAIN PARAMETERS OF THE EFFLUX (*)
(*) RESULTING FROM THE PARASITIC HEAT LOAD ON A HELIUM DEWAR IN (*)
(*) SPACE. THE SHIELD SYSTEM CONSISTS OF CONSTANT TEMPERATURE (*)
(*) SHIELDS WITH A GIVEN DIAMETER AND PIPE LENGTHS. THE MASS FLOW (*)
(*) AND SHIELD HEAT EXCHANGE ARE THEN CALCULATED ALONG WITH THE FLOW (*)
(*) VARIABLES. (*)

LABEL 14;

CONST

PI=3.141592653;
R=2077; (*J/KG-K*)
K=1.67;
CP=5200;

TYPE

DIVISIONS=ARRAY(0..20001 OF REAL;
COUNT=ARRAY(0..201 OF REAL;
ICOUNT=ARRAY(0..201 OF INTEGER;
REALOUT=FILE OF REAL;
INTOUT=FILE OF INTEGER;

VAR

```
NUMWALLS,NUMDIV,M,NTST,PKEY:INTEGER;
I,LOOPCOUNT,TAB,PIPECNT,WALLCNT,SPACECNT:INTEGER;
EFFOUTH,POUT:TEXT;
EFFINM1:REALOUT;
EFFINM2:INTOUT;
P,T,RHO:DIVISIONS;
I,QX,M,PO,TO,Q,RE,F,NU,MDOCT:DIVISIONS;
EFF,TW,LSPACE,LPIPE,LWALL,QSHIELD:COUNT;
L,LTOTAL,DIA,LOWBOUND,HIBOUND:REAL;
TOSAVE,QSAVE,MDOCT1,MDOCTMAX,MAXEFF:REAL;
```

M1,T01,P01,T1,P1,XH01,F1,RE1,Q1,QX1,MU1:REAL;

```

PROCEDURE INDATA(INDEX:INTEGER);
BEGIN
  IF INDEX = 0 THEN
    BEGIN
      (* READ IN DATA *)
      WRITE('INPUT DATA AS PROMPTED BY TERMINAL');
      WRITE(' ');
      WRITE('WHEN ASKED FOR A SERIES OF DATA SUCH AS THE');
      WRITE('SHIELD TEMPERATURES, INPUT THE DATA IN');
      WRITE('ORDER FROM THE DEWAR TO THE OUTER BOUNDARY');
      WRITE(' ');
      WRITE('ENTER THE NUMBER OF RADIATION SHIELDS');
      READLN;
      READ(NUMWALLS);
      WRITE(EFFINM1,NUMWALLS);
      WRITE('NUMBER OF SHIELDS=',NUMWALLS:5);
      WRITE(' ');
      WRITE('ENTER NUMBER OF SECTIONS FOR SOLUTION');
      READLN;
      READ(NUMDIV);
      WRITE(EFFINM2,NUMDIV);
      WRITE('NUMBER OF SOLUTION SECTIONS=',NUMDIV:5);
      WRITE(' ');
      WRITE('ENTER DIAMETER (M)');
      READLN;
      READ(DIA);
      WRITE(EFFINM1,DIA);
      WRITE('DIAMETER (M)=' ,DIA:15:7);
      WRITE(' ');
      WRITE('INPUT DISTANCES BETWEEN SHIELDS (M)');
      FOR I:=1 TO NUMWALLS+1 DO
        BEGIN
          READLN;
          READ(LSPACE(I));
          WRITE(EFFINM1,LSPACE(I));
          END;(*FOR*)
      WRITE('ENTER PIPE LENGTHS ALONG SHIELDS (M)');
      FOR I:=1 TO NUMWALLS DO
        BEGIN
          READLN;
          READ(LWALL(I));
          WRITE(EFFINM1,LWALL(I));
          END;(*FOR*)
      WRITE('ENTER TEMPERATURES OF RADIATION SHIELDS (K)');
      FOR I:=1 TO NUMWALLS DO
        BEGIN
          READLN;
          READ(TW(I));
          WRITE(EFFINM1,TW(I));
          END;(*FOR*)
      WRITE('ENTER MAXIMUM SHIELD EXCHANGER EFFECTIVENESS');
      READLN;
      READ(MAXIEFF);
      WRITE(EFFINM1,MAXIEFF);
      WRITE('MAXIMUM SHIELD EFFECTIVENESS=',MAXIEFF:15:9);
      WRITE(' ');
      WRITE('ENTER INITIAL STAGNATION T (K)');
      READLN;
      READ(T0(I));
      WRITE(EFFINM1,T0(I));
      WRITE('INITIAL STAGNATION TEMPERATURE (K)=' ,T0(I):15:9);
      WRITE(' ');
      TW(0):=T0(1);
      WRITE('ENTER INITIAL STAGNATION P (PA)');
      READLN;
      READ(P0(I));
      WRITE(EFFINM1,P0(I));
      WRITE('INITIAL STAGNATION PRESSURE (PA)=' ,P0(I):15:9);
      WRITE(' ');
      WRITE('ENTER MASS FLOW APPROXIMATION (KG/S)');
      READLN;
      READ(MDOT(I));
      WRITE(EFFINM1,MDOT(I));
      WRITE('MASS FLOW (KG/S)=' ,MDOT(I):15:9);
      WRITE(' ');
      WRITE('ENTER NUMBER OF STEPS IN RUNGE-KUTTA METHOD');
    
```

```

READLN;
READ(N);
WRITE(EFFINM2,N);
WRITELN('NUMBER OF STEPS IN RUNGE-KUTTA=',N:5);
END(*IF*)

ELSE
(* READ FILE *)
BEGIN
RESET(EFFINM1);
RESET(EFFINM2);
READ(EFFINM2,NUMWALLS);
READ(EFFINM2,NUMDIV);
READ(EFFINM1,DIA);
FOR I:=1 TO NUMWALLS+1 DO READ(EFFINM1,LSPACE(I));
FOR I:=1 TO NUMWALLS DO READ(EFFINM1,LWALL(I));
FOR I:=1 TO NUMWALLS DO READ(EFFINM1,TW(I));
READ(EFFINM1,MAXEFF);
READ(EFFINM1,TO(I));
TW(0):=TO(1);
READ(EFFINM1,PO(I));
READ(EFFINM1,MDOT(I));
READ(EFFINM2,N);
END(*ELSE*)
END(* INDATA *)

```

PROCEDURE OUTDATA;

```

BEGIN
WRITELN(EFFOUTH,'LAMFLOW OUTPUT':60);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,'VALUES OF INPUT':60);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,'NUMBER OF SECTIONS =' :50,NUMDIV:5);
WRITELN(EFFOUTH,'DEWAR STAGNATION TEMPERATURE (K)=' :50,TO(1):20);
WRITELN(EFFOUTH,'DEWAR STAGNATION PRESSURE (PA)=' :50,PO(1):20);
WRITELN(EFFOUTH,'MASS FLOW APPROXIMATION (KG/S)=' :50,MDOT(1):20);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,'SHIELD TEMPERATURES (K)':60);
FOR I:=1 TO NUMWALLS DO WRITELN(EFFOUTH,'SHIELD TEMPERATURE':40,I:2,'=' :1,TW(I):15:9);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,'VALUES OF OUTPUT':60);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,'MASS FLOW (KG/S)=' :50,MDOT(LOOPCOUNT):15:9);
WRITELN(EFFOUTH,'TOTAL HEAT TRANSFER (W)=' :50,QX(NUMDIV):15:9);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,'SHIELD HEAT TRANSFER (W)':60);
FOR I:=1 TO NUMWALLS DO WRITELN(EFFOUTH,'SHIELD HEAT (W)':40,I:2,'=' :1,QSHIELD(I):15:9);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,'SHIELD HEAT EXCHANGER EFFECTIVENESS':60);
FOR I:=1 TO NUMWALLS DO WRITELN(EFFOUTH,'SHIELD EFFECTIVENESS':40,I:2,'=' :1,EFF(I):15:9);
WRITELN(EFFOUTH,' ');
WRITELN(EFFOUTH,' ');
WRITELN('FOR AN OUTPUT OF SECTION FLOW VARIABLES, INPUT 1. OTHERWISE INPUT 0. ');
READLN;
READ(PKEY);
IF PKEY = 1 THEN
BEGIN
WRITE(EFFOUTH,'LENGTH':10,'MACH NUMBER':15,'STAG TEMP':15,'STAG PRES':15);
WRITE(EFFOUTH,'TEMP':10,'PRESSURE':10,'NUSSOLT ':10,'REYNOLD ':12);
WRITELN(EFFOUTH,'SECTION HEAT':15,'TOTAL HEAT':15);
WRITELN(EFFOUTH,' ');
FOR I:=1 TO NUMDIV DO
BEGIN
WRITE(EFFOUTH,' ':2,X(I):8:3,' ':5,M(I):10:8,' ':5,TO(I):10:6);
WRITE(EFFOUTH,' ':5,PO(I):10:3,' ':2,T(I):8:4,' ':2,PE(I):8:3);
WRITE(EFFOUTH,' ':2,NU(I):8:3,' ':3,RE(I):10:5,' ':5,Q(I):10:7);
WRITELN(EFFOUTH,' ':5,QX(I):10:7);
END(*FOR*)
END(*IF*)
WRITELN(' ');
WRITELN('FOR A PLOT FILE ENTER 1, OTHERWISE ENTER 0');
READLN;READ(TAB);
IF TAB = 0 THEN
BEGIN
WRITELN('FOR PLOT OF STAGNATION TEMPERATURE ENTER 1. ');

```

```

WRITELN('FOR PLOT OF STAGNATION PRESSURE ENTER 2. ');
WRITELN('FOR PLOT OF HEAT TRANSFER ENTER 3. ');
READLN; READ(TAB);
FOR I:=1 TO NUMDIV DO
  BEGIN
    CASE TAB OF
      1: WRITELN(POUT,T0(I));
      2: WRITELN(POUT,P0(I));
      3: WRITELN(POUT,QX(I));
    END(*CASE*)
  END(*FOR*)
END(*IF*)
WRITELN('FOR ITERATION INFORMATION, INPUT 1. OTHERWISE INPUT 0. ');
READLN;
READ(PKEY);
IF PKEY = 1 THEN
  FOR I:=1 TO LOOPCOUNT DO
    WRITELN(EFFOUTH,'INTERATION=':20,1:3,'MASS FLOW=':20,MDOT(I):15);
  WRITELN('OUTPUT STORED IN FILE EFFOUTH');
END(* OUTDATA *)

```

```

FUNCTION PWR(N,P:REAL):REAL;
(* SOLVES FOR N**P *)
VAR A:REAL;
BEGIN
  A:=P*LN(N);
  PWR:=EXP(A);
END(* PWR *)

```

```

FUNCTION B(N:REAL):REAL;
(* SOLVES OFTEN OCCURRING FUNCTION *)
BEGIN
  B:=1.0+(K-1.0)*SQR(N)/2.0;
END(* B *)

```

```

FUNCTION TEMP(T0,N:REAL):REAL;
(* CALCULATES LOCAL TEMPERATURE *)
BEGIN
  TEMP:=T0/B(N);
END(* TEMP *)

```

```

FUNCTION PRES(P0,N:REAL):REAL;
(* CALCULATES LOCAL PRESSURE *)
VAR A,C:REAL;
BEGIN
  C:=B(N);
  A:=K/(1-K);
  PRES:=P0*PWR(C,A);
END(* PRES *)

```

```

FUNCTION DENSE(P,T:REAL):REAL;
(* CALCULATES LOCAL DENSITY *)
BEGIN
  DENSE:=P/(R*T);
END(* DENSE *)

```

```

FUNCTION MU(T:REAL):REAL;
VAR A,B,C,D:REAL;
BEGIN(*MU*)
  A:=1.10419265531E-04;
  B:=1.14867759448E-07*T;
  C:=-3.222922924062E-10*SQR(T);
  D:=4.725700293922E-13*T*SQR(T);
  MU:=A+B+C+D;
END(*MU*)

```

```

FUNCTION COND(T:REAL):REAL;
VAR A,B,C,D:REAL;
BEGIN(*CONDUCTIVITY CALCULATION*)
  A:=0.0157218071;

```

```

B:=4.3416525E-04*T;
C:=-4.99407521E-07*SQR(T);
D:=-4.32003094E-10*T*SQR(T);
COND:=A+B+C+D;
END;(*COND*)

```

```

FUNCTION PRANDTL(T:REAL):REAL;
  VAR A,B,C,D,E,F:REAL;
  BEGIN(*PRANDTL NUMBER CALCULATION*)
    A:=8.570232794;
    B:=8.0055072291*T;
    C:=-7.50324532E-05*SQR(T);
    D:=-4.32720501E-07*T*SQR(T);
    E:=-1.11661445E-09*SQR(T)*SQR(T);
    F:=1.07403042E-12*T*SQR(T)*SQR(T);
    PRANDTL:=A+B+C+D+E+F;
  END;(*PRANDTL*)

```

```

FUNCTION MFRIC(M1,M2,D,F,L:REAL):REAL;
  VAR MFR:REAL;
  (* USED IN CASE 2 OF NEWTON *)
  BEGIN
    MFR:=(SQR(M2)-SQR(M1))/(K*SQR(M1*M2))+(K+1.0)/(2.0*K)*LN(B(M2)*SQR(M1)/(B(M1)*SQR(M2)))-4.0*F*L/D;
    MFRIC:=MFR;
  END;(* MFRIC *)

```

```

FUNCTION DMFRIC(M1,M2:REAL):REAL;
  (* USED IN CASE 2 OF NEWTON *)
  VAR DMF:REAL;
  BEGIN
    DMF:=2.0/(K*M2*SQR(M2))+(K+1)/K*M2*SQR(B(M1))/(PWR(M1,4)*PWR(B(M2),3));
    DMFRIC:=DMF;
    IF DMF < 1.274 THEN (*FUNCTION UNSTABLE, MACH=1 APPROACHED*)
      DMFRIC:=1.0;
    END;(* DMFRIC *)

```

```

FUNCTION MFRST(M,D,MDOOT,P0,T0:REAL):REAL;
  (* USED IN CASE 3 OF NEWTON *)
  VAR A,C:REAL;
  BEGIN
    A:=B(M);
    C:=(K+1.0)/(2.0*(1.0-K));
    MFRST:=(M*P0*PI*SQR(D)/4.0)*SQRT(K/(R*T0))*PWR(A,C)-MDOOT;
  END;(* MFRST *)

```

```

FUNCTION DMFRST(M,D,P0,T0:REAL):REAL;
  (* USED IN CASE 3 OF NEWTON *)
  VAR A,C,E:REAL;
  BEGIN
    A:=B(M);
    C:=(K+1.0)/(2.0*(1.0-K));
    E:=(3.0*K-1.0)/(2.0*(1.0-K));
    DMFRST:=(PI*SQR(D)/4.0)*P0*SQRT(K/(R*T0))*(PWR(A,C)-SQR(M)*(K+1.0)/2.0*PWR(A,E));
  END;(* DMFRST *)

```

```

FUNCTION NEWTON(KEY:INTEGER; M1,D,F,L,LTOT,P0,T0:REAL):REAL;
  VAR MNEW,MOLD,DIFF,D1,D2,A,MDOOT,DMF:REAL;
  COUNT:INTEGER;
  BEGIN(* NEWTON *)
    DIFF:=1.0;
    CASE KEY OF
      1: BEGIN
        (* CALCULATES EXIT MACH FOR SECTION WITH FRICTION ONLY *)
        MOLD:=1.01*M1;
        COUNT:=1;
        WHILE (DIFF) > 0.00005 AND (COUNT < 10) DO
          BEGIN
            DMF:=DMFRIC(M1,MOLD);
            MNEW:=MOLD-MFRIC(M1,MOLD,D,F,L)/DMF;
            DIFF:=ABS(MOLD-MNEW);
            IF DMF = 1.0 THEN (*FUNCTION UNSTABLE, EXIT LOOP*)

```

```

      BEGIN
      DIFF:=0.00001;
      MNEW:=1.0;
      END;(*IF*)
      COUNT:=COUNT+1;
      MOLD:=MNEW;
      END(* WHILE *)
    END;(*CASE 1*)
  2: BEGIN
    (* CALCULATES INLET MACH NUMBER *)
    MDOT:=L;
    MOLD:=M1;
    WHILE (DIFF) = 0.00001 DO
      BEGIN
        MNEW:=MOLD-MFIRST(MOLD,D,MDOT,P0,T0)/DMFIRST(MOLD,D,P0,T0);
        DIFF:=ABS(MOLD-MNEW);
        MOLD:=MNEW;
        END(* WHILE *)
      END(* 2 *)
    END;(* CASE *)
    NEWTON:=MOLD;
    END;(* NEWTON *)

```

```

PROCEDURE INLETMACH(D,MDOT,P0,T0:REAL; VAR M,T,P,RHO:REAL);
  (* CALCULATES INLET CONDITIONS *)
  VAR A:REAL;
  BEGIN(* INLETMACH *)
    M:=NEWTON(2,M,D,0,MDOT,0.0,P0,T0);
    T:=TEMP(T0,M);
    P:=PRES(P0,M);
    RHO:=DENSE(P,T);
  END;(* INLETMACH *)

```

```

PROCEDURE FRICTION(T0,DIA,F,L:REAL;VAR M,T,P,RHO,P0,Q:REAL);
  (* FOR FLOW WITH FRICTION ONLY *)
  VAR A,C,NEWM:REAL;
  BEGIN
    NEWM:=NEWTON(1,M,DIA,F,L,LTOTAL,0.0);
    A:=B(NEWM)/B(M);
    C:=(K+1.0)/(2.0*(K-1.0));
    P0:=P0*M/NEWM*FVR(A,C);
    M:=NEWM;
    T:=TEMP(T0,M);
    P:=PRES(P0,M);
    RHO:=DENSE(P,T);
    Q:=0;
  END;(*FRICTION*)

```

```

FUNCTION STAGTEMP(T0,TV,NU,KF,MDOT,L:REAL):REAL;
  (* CALCULATES NEW STAGNATION TEMPERATURE FOR FLOW WITH HEAT TRANSFER *)
  BEGIN
    STAGTEMP:=TV+((T0-TV)/EXP((NU*PI*KF*L)/(MDOT*CP)));
  END;(*STAGTEMP*)

```

```

FUNCTION RFUNC(M,T0,TV,PR,NU:REAL; KEY:INTEGER):REAL;
  (* SOLVES AN OFTEN USED FUNCTION IN PROCEDURE HEATTRANS *)
  VAR A,C,D,E:REAL;
  BEGIN
    A:=M*B(M)/(2.0*(1.0-SQR(M)));
    C:=(1.0+K*SQR(M))/T0;
    D:=K*SQR(M);
    IF KEY = 1 THEN
      BEGIN
        E:=2.0/(TV-T0);
        END(*IF*)
      ELSE
        BEGIN
          E:=16.0*PR/(NU*(TV-T0));
          END(*ELSE*)
        RFUNC:=A*(D+E*C);
      END;(* RFUNC *)

```

```

PROCEDURE HEATTRANS(TV,L,D,MDOT:REAL; N:INTEGER; VAR M,T0,P0,T,P,RHO,KE,Y,Q,QX,MU:REAL);
(* CALCULATES FLOW VARIABLES FOR FLOW WITH FRICTION AND HEAT TRANSFER *)
VAR
  C1,C2,C3,C4,STEP,A,C,NEWL,FACTOR:REAL;
  T01,T02,M1,M2,PR,KF,DELN:REAL;
  KEY,I,TEST:INTEGER;
BEGIN
  KEY:=0;
  TEST:=0;
  (* FLAG KEY IF FLOW IS TURBULENT *)
  IF MU > 3.44 THEN KEY:=1;
  (* CALCULATE NEW STAGNATION TEMPERATURE *)
  PR:=PRANDTL(T);
  KF:=COND(T);
  T01:=T0;
  T02:=STAGTEMP(T01,TV,MU,KF,MDOT,L);
  IF (TV-T02) < 0.0001 THEN
    (* WALL TEMPERATURE APPROACHED, EXIT PROGRAM *)
    BEGIN
      A:=(MDOT*CP)/(PI*MU*KF);
      NEWL:=A*LN(0.03);
      FACTOR:=L/NEWL;
      WRITELN('THE FLUID TEMPERATURE APPROACHES THE WALL TEMPERATURE');
      WRITELN('AT AN INDETERMINATE LENGTH. INCREASE THE NUMBER OF');
      WRITELN('SOLUTION SECTIONS. SUGGESTED INCREASE BY FACTOR OF',FACTOR:15,',',1);
      WRITELN('OR MORE. ');
      GOTO 10;
    END;(*IF*)
  (* STEPWISE SOLUTION USING RUNGE-KUTTE METHOD *)
  (* CALCULATE STEP SIZE *)
  STEP:=ABS((T02-T01)/N);
  M1:=M;
  I:=1;
  WHILE (I <= N) AND (TEST = 0) DO
    BEGIN
      C1:=RFUNC(M,T0,TV,PR,MU,KEY);
      IF C1 < 0 THEN C1:=0;(*FUNCTION UNSTABLE*)
      C2:=RFUNC((M+C1*STEP/2.0),(T0+STEP/2.0),TV,PR,MU,KEY);
      IF C2 < 0 THEN C2:=0;(*FUNCTION UNSTABLE*)
      C3:=RFUNC((M+C2*STEP/2.0),(T0+STEP/2.0),TV,PR,MU,KEY);
      IF C3 < 0 THEN C3:=0;(*FUNCTION UNSTABLE*)
      C4:=RFUNC(M,C3*STEP,T0+STEP,TV,PR,MU,KEY);
      IF C4 < 0 THEN C4:=0;(*FUNCTION UNSTABLE*)
      DELN:=STEP/6.0*(C1+2.0*C2+2.0*C3+C4);
      M:=M+DELN;
      T0:=T0+STEP;
      I:=I+1;
      IF M > 1 THEN (*FUNCTION UNSTABLE, EXIT LOOP*)
        BEGIN
          TEST:=1;
          M:=1.0;
          END;(*IF*)
        END;(*WHILE*)
      IF (C1 = 0) OR (C2 = 0) OR (C3 = 0) OR (C4 = 0) THEN
        (*FUNCTION UNSTABLE, EXIT LOOP*)
        BEGIN
          TEST:=1;
          M:=1;
          END;(*IF*)
    END;
  M2:=M;
  A:=R(M2)/R(M1);
  C:=(K+1.0)/(2.0*(K-1.0));
  P0:=P0*(M1/M2)*SQRT(T02/T01)*PWR(A,C);
  P:=PRES(P0,M2);
  T:=TEMP(T02,M2);
  RHO:=DENSE(P,T);
  RE:=4.0*MDOT/(PI*D*MU(T));
  IF RE <= 2000 THEN
    BEGIN (* LAMINAR FLOW *)
      F:=16.0/RE;
      MU:=3.44;
      END;(*IF*)
  ELSE
    BEGIN (* TURBULENT FLOW *)
      F:=0.046/PWR(RE,0.2);
      MU:=0.023*PWR(RE,0.8)*PWR(PR,0.4);
      END;(*ELSE*)
  Q:=CP*MDOT*(T02-T01);
  QX:=QX+Q;

```

END;(* HEATTRANS *)

PROCEDURE SECTION(MWALL:INTEGER; VAR NDIV:INTEGER; LS,LV:COUNT; VAR I:DIVISIONS);
(* THIS PROCEDURE DIVIDES THE PIPE LENGTHS INTO NONOVERLAPPING SECTIONS *)

```

VAR
  RESL,LTOT,L:REAL;
  COUNT1,COUNT2,I,J,KEY:INTEGER;

BEGIN (* SECTION *)
  (* CALCULATE NOMINAL SECTION SIZE *)
  LTOT:=0;
  FOR I:=1 TO MWALL DO LTOT:=LTOT+LV(I);
  FOR I:=1 TO MWALL+1 DO LTOT:=LTOT+LS(I);
  L:=LTOT/NDIV;
  (* CALCULATE PIPE SECTIONS *)
  LTOT:=0;
  I(0):=0;
  J:=0;
  KEY:=0;
  COUNT1:=1;
  COUNT2:=1;
  FOR I:=1 TO 1*MWALL+1 DO
    BEGIN
      KEY:=KEY+1;
      IF KEY = 1 THEN
        BEGIN
          LTOT:=LTOT+LS(COUNT1);
          COUNT1:=COUNT1+1;
          END(*IF*)
        ELSE
          BEGIN
            LTOT:=LTOT+LV(COUNT2);
            COUNT2:=COUNT2+1;
            KEY:=0;
            END(*ELSE*)
          WHILE I(J) < LTOT DO
            BEGIN
              J:=J+1;
              I(J):=I(J-1)+L;
              END(*WHILE*)
            IF I(J) > LTOT THEN
              BEGIN
                RESL:=LTOT-I(J-1);
                I(J):=I(J-1)+RESL;
                END(*IF*)
              END;(*FOR*)
            NDIV:=J;
            I(NDIV+1):=LTOT+1;
            END;(*SECTION*)
  
```

```

PROCEDURE SETBOUND(KEY:INTEGER; VAR HIB,LOWB,MDOT:REAL);
  BEGIN(*SETBOUND*)
    IF KEY = 1 THEN
      BEGIN
        LOWB:=MDOT;
        END(*IF*)
      ELSE
        BEGIN
          HIB:=MDOT;
          END(*ELSE*)
        MDOT:=(2.0*LOWB+HIB)/3.0;
        END;(*SETBOUND*)
  
```

```

PROCEDURE INREASSIGN;
  (*DEFINES VALUES FOR USE IN PARAMETER LIST *)
  BEGIN(*INREASSIGN*)
    L:=I(1)-I(1)-1;
    M1:=M(1);
    T01:=T(1);
    P01:=P(1);
    T1:=T(1);
    P1:=P(1);
    RHO1:=RHO(1);
  
```


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F1:=F(1);
RE1:=RE(1);

NU1:=NU(1);
Q1:=Q(1);
QX1:=QX(1);
END;(*INREASSIGN*)

```

```

PROCEDURE OUTREASSIGN;
(* DEFINES NEW VALUES OBTAINED FROM PARAMETER LIST *)
BEGIN(*OUTREASSIGN*)
  I:=1;
  NI1:=NI;
  TO1:=TO;
  PO1:=PO;
  TI1:=TI;
  PI1:=PI;
  RHQ1:=RHQ;
  F1:=F;
  RE1:=RE;
  NU1:=NU;
  Q1:=Q;
  QX1:=QX;
END;(*OUTREASSIGN*)

```

```

PROCEDURE INITIALIZE;
(* PREPARES VARIABLES FOR USE IN MAIN PROGRAM *)
VAR
  JMTIGER;
  A,C:REAL;
BEGIN(*INITIALIZE*)
  MDOT(1):=0;
  (*DETERMINE MAXIMUM MASS FLOW*)
  A:=1.0+0.405*(K-1.0);
  C:=(K+1.0)/(2.0*(1.0-K));
  MDOTMAX:=(PI*SQRT(DIA)/4.0)*PO(1)*SQRT(K/(R*TO(1)))*PVR(A,C);
  (*CHECK GIVEN MASS FLOW AND REDUCE IF NECESSARY*)
  IF MDOT(1) > MDOTMAX THEN MDOT(1):=0.7*MDOTMAX;
  (* CALCULATE TOTAL PIPE LENGTH *)
  LTOTAL:=0;
  FOR I:=1 TO NUMVALLS DO LTOTAL:=LTOTAL+LWALL(I);
  FOR I:=1 TO NUMVALLS+1 DO LTOTAL:=LTOTAL+LSPACE(I);
  (*CALCULATE SECTION LENGTHS*)
  (*INITIALIZE SECTION ARRAY*)
  SECTION(NUMVALLS,NUMDIV,LSPACE,LWALL,X);
  (*START LOOP COUNTER*)
  LOOPCOUNT:=0;
  (*INITIAL MACH NUMBER GUESS*)
  NI1:=0.01;
  (*INITIALIZE END MACH NUMBER SO THAT IT MAY BE USED FOR TEST*)
  MINUMDIV:=0.0;
  (*INITIALIZE BOUNDS *)
  RHBOUND:=MDOTMAX;
  LOWBOUND:=0.0;
END;(*INITIALIZE*)

```

```

PROCEDURE REINITIAL;
(* PREPARES VARIABLES FOR USE IN MAIN PROGRAM *)
VAR J:INTEGER;
BEGIN(*REINITIAL*)
  (*INITIALIZE COUNTERS*)
  I:=1;
  PIPECNT:=1;
  WALLCNT:=1;
  SPACECNT:=1;
  (*INCREMENT LOOP COUNTER*)
  LOOPCOUNT:=LOOPCOUNT+1;
  (*CALCULATE ENTRANCE CONDITIONS*)
  LPIPE(I):=LSPACE(I);
  QX(I):=0;Q(I):=0;
  RE(I):=4.0*MDOT(LOOPCOUNT)/(PI*NU(TO(I))*DIA);
  IF RE(I) <= 2000 THEN
    BEGIN (* LAMINAR FLOW *)
      F(I):=16.0/RE(I);
      NU(I):=3.46;
    END(*IF*)
  ELSE

```

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```

      BEGIN (* TURBULENT FLOW *)
      FC11:=0.046/PWR(RE11,0.2);
      MU11:=0.023*PWR(RE11,0.8)*PWR(PRANDTL(T011),0.4);
      END;(*ELSE*)
      (*CALCULATE INLET CONDITIONS*)
      INLETHACH(DIA,MDOT1,LOOPCOUNT),P011,T011,M11,T11,P11,RHO11);
      (*INITIALIZE CONTROL CARD*)
      MTEST:=1;
      END;(*REINITIAL*)

10: BEGIN(* MAIN PROGRAM *)
      WRITELN('TO ENTER DATA, ENTER 1. OTHERWISE, ENTER 0. ');
      READLN;
      READ(TAB);
      INDATA(TAB);
      INITIALIZE;

      REPEAT (* UNTIL THE FLOW IS CHOKED *)
      REINITIAL;
      WHILE (1 < NUMDIV) AND (M11 <= 0.97) DO
      BEGIN
      (* BEGIN FRICTION FLOW CALCULATIONS *)
      WHILE (X11 <= LPIPE1PIPECNT) AND (M11 <= 0.97) DO
      BEGIN
      INREASSIGN;
      FRICTION(T01,DIA,P1,L,M1,T1,P1,RHO1,P01,Q1);
      OUTREASSIGN;
      END;(*WHILE*)
      IF WALLCNT <= NUMWALLS THEN
      BEGIN
      PIPECNT:=PIPECNT+1;
      SPACECNT:=SPACECNT+1;
      LPIPE1PIPECNT:=LPIPE1PIPECNT+LWALL(WALLCNT);
      END;(*IF*)
      (* BEGIN HEAT TRANSFER CALCULATIONS *)
      TOSAVE:=T011;
      EFF1WALLCNT:=0;
      QSAVE:=QX11;
      WHILE (EFF1WALLCNT < MAXEFF) AND (X11 < LPIPE1PIPECNT) AND (M11 <= 0.97) DO
      BEGIN
      INREASSIGN;
      HEATTRANS(TW1WALLCNT,L,DIA,MDOT1,LOOPCOUNT),N,M1,T01,P01,T1,P1,RHO1,RE1,P1,Q1,QX1,MU1);
      OUTREASSIGN;
      EFF1WALLCNT:=(T011-TOSAVE)/(TW1WALLCNT-TOSAVE);
      END;(*WHILE*)
      QSHIELD(WALLCNT):=QX11-QSAVE;
      IF WALLCNT <= NUMWALLS THEN
      BEGIN
      PIPECNT:=PIPECNT+1;
      WALLCNT:=WALLCNT+1;
      LPIPE1PIPECNT:=LPIPE1PIPECNT+LSPACE1SPACECNT;
      END;(*IF*)
      END;(*WHILE*)
      IF NUMDIV > 1 THEN MTEST:=2;
      MDOT1:=MDOT1/LOOPCOUNT1;
      SETBOUND(MTEST,HIBOUND,LOWBOUND,MDOT1);
      MDOT1/LOOPCOUNT+1:=MDOT1;
      UNTIL (MINUMDIV > 0.97) AND (MINUMDIV > 1.03) OR (LOOPCOUNT = 30);

      OUTDATA;
10: END;(* MAIN PROGRAM *)

```

SYNTH.
TO ENTER DATA, ENTER 1. OTHERWISE, ENTER 0.
? 1
INPUT DATA AS PROMPTED BY TERMINAL

WHEN ASKED FOR A SERIES OF DATA SUCH AS THE
SHIELD TEMPERATURES, INPUT THE DATA IN
ORDER FROM THE DEWAR TO THE OUTER BOUNDARY

ENTER THE NUMBER OF RADIATION SHIELDS
? 5
NUMBER OF SHIELDS= 5

ENTER NUMBER OF SECTIONS FOR SOLUTION
? 125
NUMBER OF SOLUTION SECTIONS= 125

ENTER DIAMETER (M)
? 0.0016
DIAMETER (M)= 0.0016000

INPUT DISTANCES BETWEEN SHIELDS (M)
? 0.019
? 0.021
? 0.024
? 0.026
? 0.034
? 0.035
ENTER PIPE LENGTHS ALONG SHIELDS (M)
? 0.032
? 0.01
? 0.008
? 0.006
? 0.038
ENTER TEMPERATURES OF RADIATION SHIELDS (K)
? 21.7
? 39.7
? 65.4
? 101.1
? 146.6
ENTER MAXIMUM SHIELD EXCHANGER EFFECTIVENESS
? 0.98
MAXIMUM SHIELD EFFECTIVENESS= 0.980000000

ENTER INITIAL STAGNATION T (K)
? 4.23
INITIAL STAGNATION TEMPERATURE (K)= 4.230000000

ENTER INITIAL STAGNATION P (PA)
? 2100.0
INITIAL STAGNATION PRESSURE (PA)= 2100.000000000

ENTER MASS FLOW APPROXIMATION (KG/S)
? 0.00000163
MASS FLOW (KG/S)= 0.000001630

ENTER NUMBER OF STEPS IN RUNGE-KUTTA METHOD
? 4
NUMBER OF STEPS IN RUNGE-KUTTA= 4
FOR AN OUTPUT OF SECTION FLOW VARIABLES, INPUT 1. OTHERWISE INPUT 0.
? 1

FOR A PLOT FILE ENTER 1, OTHERWISE ENTER 0
? 0
FOR ITERATION INFORMATION, INPUT 1. OTHERWISE INPUT 0.
? 1
OUTPUT STORED IN FILE EFFOUTM
1.824 CP SECS, 101466B CM USED.
/

TYPE EFFOUTH

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LAMFLOW OUTPUT

VALUES OF INPUT

NUMBER OF SECTIONS = 128
DEWAR STAGNATION TEMPERATURE (K) = 4.23000000000E+000
DEWAR STAGNATION PRESSURE (PA) = 2.10000000000E+003
MASS FLOW APPROXIMATION (KG/S) = 1.63000000000E-004

SHIELD TEMPERATURES (K)

SHIELD TEMPERATURE 1= 21.700000000
SHIELD TEMPERATURE 2= 39.700000000
SHIELD TEMPERATURE 3= 65.400000000
SHIELD TEMPERATURE 4= 101.100000000
SHIELD TEMPERATURE 5= 146.400000000

VALUES OF OUTPUT

MASS FLOW (KG/S) = 0.000000058
TOTAL HEAT TRANSFER (W) = 0.630151115

SHIELD HEAT TRANSFER (W)

SHIELD HEAT (W) 1= 0.045715165
SHIELD HEAT (W) 2= 0.042899052
SHIELD HEAT (W) 3= 0.071678217
SHIELD HEAT (W) 4= 0.095742199
SHIELD HEAT (W) 5= 0.354116483

SHIELD HEAT EXCHANGER EFFECTIVENESS

SHIELD EFFECTIVENESS 1= 0.443247782
SHIELD EFFECTIVENESS 2= 0.463717702
SHIELD EFFECTIVENESS 3= 0.434301394
SHIELD EFFECTIVENESS 4= 0.380185874
SHIELD EFFECTIVENESS 5= 0.986241974

LENGTH	MASS NUMBER	STAG TEMP	STAG PRES	TEMP	PRESSURE	NUSSELT #	REYNOLD #	SECTION HEAT	TOTAL HEAT
0.00202	0.01473847	4.230000	2100.00000	4.2297	2099.419	3.660	429.50017	0.0000000	0.0000000
0.00405	0.01478746	4.230000	2093.04477	4.2297	2091.663	3.660	429.50017	0.0000000	0.0000000
0.00607	0.01483481	4.230000	2084.11226	4.2297	2085.729	3.660	429.50017	0.0000000	0.0000000
0.00810	0.01488614	4.230000	2079.20237	4.2297	2078.818	3.660	429.50017	0.0000000	0.0000000
0.01012	0.01493542	4.230000	2072.31586	4.2297	2071.929	3.660	429.50017	0.0000000	0.0000000
0.01214	0.01498528	4.230000	2065.45022	4.2297	2065.063	3.660	429.50017	0.0000000	0.0000000
0.01417	0.01503510	4.230000	2058.60788	4.2297	2058.219	3.660	429.50017	0.0000000	0.0000000
0.01619	0.01508509	4.230000	2051.78772	4.2297	2051.398	3.660	429.50017	0.0000000	0.0000000
0.01822	0.01513525	4.230000	2044.98990	4.2297	2044.599	3.660	429.50017	0.0000000	0.0000000
0.01900	0.01518558	4.230000	2038.21427	4.2297	2037.822	3.660	429.50017	0.0000000	0.0000000
0.02102	0.01523585	4.230000	2031.49128	4.2297	2031.098	3.660	429.50017	0.0000000	0.0000000
0.02305	0.01788757	5.828338	2031.22570	5.8277	2030.683	3.660	385.73793	0.0071299	0.0871299
0.02507	0.02009720	7.355442	2030.92014	7.3546	2030.235	3.660	351.78038	0.0048131	0.1139430
0.02710	0.02198954	8.800362	2030.57312	8.7989	2029.753	3.660	324.94683	0.0044447	0.0283877
0.02912	0.02362583	10.153898	2030.18408	10.1520	2029.238	3.660	303.43258	0.0040379	0.0244254
0.03114	0.02505172	11.410614	2029.75284	11.4082	2028.698	3.660	285.98168	0.0036068	0.0320316
0.03317	0.02629846	12.567686	2029.28046	12.5648	2028.109	3.660	271.69305	0.0031615	0.0371932
0.03519	0.02739014	13.624788	2028.76818	13.6214	2027.498	3.660	259.90346	0.0027154	0.0417087
0.03722	0.02834634	14.583734	2028.21784	14.5790	2026.858	3.660	250.11411	0.0024777	0.0441864
0.03924	0.02918360	15.448058	2027.62165	15.4436	2026.190	3.660	241.94290	0.00238554	0.0508420
0.04126	0.02991624	16.222553	2027.01205	16.2177	2025.498	3.660	235.09248	0.00234549	0.0534970
0.04329	0.03055680	16.912962	2026.36161	16.9077	2024.783	3.660	229.32831	0.00230798	0.0565748
0.04531	0.03111639	17.525545	2025.68296	17.5199	2024.046	3.660	224.46321	0.00227326	0.0592094
0.04734	0.03160489	18.064828	2024.97871	18.0608	2023.291	3.660	220.34627	0.00224156	0.0617240
0.04936	0.032083107	18.543362	2024.25138	18.5370	2022.518	3.660	216.85481	0.00221257	0.0638497
0.05100	0.03240272	18.961539	2023.50334	18.9549	2021.730	3.660	213.68828	0.00218654	0.0657152
0.05302	0.03264727	18.961539	2019.23756	18.9548	2017.461	3.660	213.88828	0.00200000	0.0657152
0.05505	0.03254168	18.961539	2014.87486	18.9548	2013.094	3.660	213.88828	0.0000000	0.0657152
0.05707	0.03241228	18.961539	2010.51919	18.9548	2008.735	3.660	213.88828	0.0000000	0.0657152
0.05910	0.03268307	18.961539	2006.17053	18.9548	2004.382	3.660	213.88828	0.0000000	0.0657152
0.06112	0.03275404	18.961539	2001.82884	18.9547	2000.037	3.660	213.88828	0.0000000	0.0657152
0.06314	0.03282524	18.961539	1997.49419	18.9547	1995.698	3.660	213.88828	0.0000000	0.0657152
0.06517	0.03289661	18.961539	1993.16447	18.9547	1991.367	3.660	213.88828	0.0000000	0.0657152
0.06719	0.03296818	18.961539	1988.84571	18.9546	1987.042	3.660	213.88828	0.0000000	0.0657152
0.06922	0.03303995	18.961539	1984.53187	18.9546	1982.724	3.660	213.88828	0.0000000	0.0657152
0.07124	0.03311192	18.961539	1980.22495	18.9546	1978.413	3.660	213.88828	0.0000000	0.0657152

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0.07300	0.03318400	18.941539	1975.92494	18.9545	1974.109	3.660	213.88820	0.0000000	0.0457152
0.07402	0.03325054	18.941539	1971.98175	18.9545	1970.162	3.660	213.88820	0.0000000	0.0457152
0.07485	0.03342057	21.732783	1970.91570	21.7235	1968.829	3.660	196.24627	0.0123617	0.0780769
0.07807	0.03744981	24.249389	1969.49945	24.2570	1967.360	3.660	182.64394	0.0113154	0.0893923
0.08010	0.03943231	26.552986	1968.33701	26.5392	1965.784	3.660	172.11285	0.0101071	0.0995795
0.08200	0.04094309	28.578330	1966.83487	28.5623	1964.085	3.660	163.88802	0.0090340	0.1086142
0.08402	0.04101142	28.578330	1963.55595	28.5622	1960.801	3.660	163.88802	0.0080009	0.1086142
0.08605	0.04108184	28.578330	1960.20630	28.5622	1957.147	3.660	163.88802	0.0080000	0.1086142
0.08807	0.04115231	28.578330	1956.85817	28.5621	1954.894	3.660	163.88802	0.0080000	0.1086142
0.09010	0.04122297	28.578330	1953.51154	28.5621	1950.742	3.660	163.88802	0.0080000	0.1086142
0.09212	0.04129384	28.578330	1950.16642	28.5620	1947.392	3.660	163.88802	0.0080000	0.1086142
0.09414	0.04136492	28.578330	1946.82279	28.5620	1944.544	3.660	163.88802	0.0080000	0.1086142
0.09617	0.04143622	28.578330	1943.48064	28.5619	1940.697	3.660	163.88802	0.0080000	0.1086142
0.09819	0.04150773	28.578330	1940.13994	28.5618	1937.352	3.660	163.88802	0.0080000	0.1086142
0.10022	0.04157946	28.578330	1936.80074	28.5618	1934.008	3.660	163.88802	0.0080000	0.1086142
0.10224	0.04165140	28.578330	1933.46301	28.5617	1930.665	3.660	163.88802	0.0080000	0.1086142
0.10426	0.04172357	28.578330	1930.12672	28.5617	1927.324	3.660	163.88802	0.0080000	0.1086142
0.10600	0.04179595	28.578330	1926.79187	28.5616	1923.984	3.660	163.88802	0.0080000	0.1086142
0.10802	0.04186469	28.578330	1923.43540	28.5616	1920.823	3.660	163.88802	0.0080000	0.1086142
0.11005	0.04193257	34.44317	1921.45571	34.4217	1918.061	3.660	144.11636	0.0261742	0.1347904
0.11207	0.04198113	39.845474	1918.80013	39.8127	1914.867	3.660	130.17586	0.0240848	0.1588752
0.11400	0.05257984	44.446622	1915.66882	44.4053	1911.254	3.660	120.14150	0.0214172	0.1802924
0.11602	0.05270749	44.446622	1911.40652	44.4051	1906.621	3.660	120.14150	0.0080000	0.1802924
0.11805	0.05283932	44.446622	1906.29624	44.4049	1901.859	3.660	120.14150	0.0080000	0.1802924
0.12007	0.05297186	44.446622	1901.54437	44.4047	1897.896	3.660	120.14150	0.0080000	0.1802924
0.12210	0.05310511	44.446622	1896.79087	44.4045	1893.332	3.660	120.14150	0.0080000	0.1802924
0.12412	0.05323908	44.446622	1892.03571	44.4043	1887.545	3.660	120.14150	0.0080000	0.1802924
0.12614	0.05337378	44.446622	1887.27886	44.4041	1882.797	3.660	120.14150	0.0080000	0.1802924
0.12817	0.05350921	44.446622	1882.52028	44.4038	1878.027	3.660	120.14150	0.0080000	0.1802924
0.13019	0.05364539	44.446622	1877.75994	44.4036	1873.255	3.660	120.14150	0.0080000	0.1802924
0.13222	0.05378231	44.446622	1872.99779	44.4034	1868.482	3.660	120.14150	0.0080000	0.1802924
0.13424	0.05391999	44.446622	1868.23381	44.4032	1863.704	3.660	120.14150	0.0080000	0.1802924
0.13626	0.05405843	44.446622	1863.46795	44.4030	1858.929	3.660	120.14150	0.0080000	0.1802924
0.13829	0.05419764	44.446622	1858.70018	44.4027	1854.149	3.660	120.14150	0.0080000	0.1802924
0.14000	0.05433763	44.446622	1853.93046	44.4025	1849.368	3.660	120.14150	0.0080000	0.1802924
0.14202	0.05447844	44.446622	1849.58354	44.4023	1845.010	3.660	120.14150	0.0080000	0.1802924
0.14405	0.05461968	55.909883	1844.87728	55.9480	1839.132	3.660	102.45910	0.0502437	0.2305361
0.14600	0.05476151	66.109399	1838.80167	66.0109	1831.982	3.660	91.05174	0.0454985	0.2760346
0.14802	0.05490371	66.109399	1831.55385	66.0101	1824.707	3.660	91.05174	0.0080000	0.2760346
0.15005	0.05504624	66.109399	1824.09879	66.0093	1817.224	3.660	91.05174	0.0080000	0.2760346
0.15207	0.05518904	66.109399	1816.62923	66.0085	1809.726	3.660	91.05174	0.0080000	0.2760346
0.15410	0.05533212	66.109399	1809.14496	66.0077	1802.213	3.660	91.05174	0.0080000	0.2760346
0.15612	0.05547542	66.109399	1801.64575	66.0068	1794.685	3.660	91.05174	0.0080000	0.2760346
0.15814	0.05561892	66.109399	1794.13138	66.0059	1787.141	3.660	91.05174	0.0080000	0.2760346
0.16017	0.05576260	66.109399	1786.60163	66.0051	1779.581	3.660	91.05174	0.0080000	0.2760346
0.16219	0.05590646	66.109399	1779.05626	66.0042	1772.006	3.660	91.05174	0.0080000	0.2760346
0.16422	0.05605045	66.109399	1771.49504	66.0033	1764.415	3.660	91.05174	0.0080000	0.2760346
0.16624	0.05619450	66.109399	1763.91772	66.0023	1756.807	3.660	91.05174	0.0080000	0.2760346
0.16826	0.05633873	66.109399	1756.32406	66.0014	1749.182	3.660	91.05174	0.0080000	0.2760346
0.17029	0.05648314	66.109399	1748.71381	66.0005	1741.540	3.660	91.05174	0.0080000	0.2760346
0.17231	0.05662772	66.109399	1741.08471	65.9995	1733.882	3.660	91.05174	0.0080000	0.2760346
0.17434	0.05677247	66.109399	1733.44250	65.9985	1726.205	3.660	91.05174	0.0080000	0.2760346
0.17636	0.05691738	66.109399	1725.78092	65.9975	1718.512	3.660	91.05174	0.0080000	0.2760346
0.17838	0.05706245	66.109399	1718.10170	65.9965	1710.800	3.660	91.05174	0.0080000	0.2760346
0.18040	0.05720768	66.109399	1710.40457	65.9955	1703.069	3.660	91.05174	0.0080000	0.2760346
0.18202	0.05735311	66.109399	1703.66744	65.9946	1695.304	3.660	91.05174	0.0080000	0.2760346
0.18405	0.05749877	86.078887	1695.99074	85.8817	1684.335	3.660	75.89737	0.0890809	0.3651153
0.18607	0.05764460	103.554686	1688.61638	103.2643	1668.894	3.660	67.10205	0.0779569	0.4438724
0.18810	0.05779065	117.378670	1663.97409	116.9976	1650.543	3.660	61.89485	0.0616667	0.5047391
0.19012	0.05793692	127.456459	1644.57728	126.9959	1629.807	3.660	58.77742	0.0449555	0.5496946
0.19214	0.05808336	134.371112	1622.99295	133.8451	1607.203	3.660	56.89448	0.0308452	0.5805398
0.19417	0.05822997	138.919405	1599.73146	138.3405	1583.160	3.660	55.75387	0.0202901	0.6088299
0.19619	0.05837680	141.829058	1575.18431	141.2040	1557.994	3.660	55.06225	0.0129786	0.6138086
0.19822	0.05852387	143.656982	1549.62479	142.9960	1531.917	3.660	54.64333	0.0081537	0.6211623
0.20024	0.05867114	144.792292	1523.22924	144.8971	1505.864	3.660	54.39047	0.0050640	0.6270271
0.20226	0.05881861	145.492608	1496.10248	144.7645	1477.510	3.660	54.23891	0.0031240	0.6301511
0.20429	0.05896627	145.992608	1468.44279	144.7364	1449.492	3.660	54.23891	0.0000000	0.6301511
0.20631	0.05911408	145.492608	1440.25988	144.7060	1420.929	3.660	54.23891	0.0000000	0.6301511
0.20834	0.05926203	145.992608	1411.52143	144.6731	1391.787	3.660	54.23891	0.0000000	0.6301511
0.21036	0.05941014	145.492608	1382.19170	144.6373	1362.027	3.660	54.23891	0.0000000	0.6301511
0.21238	0.05955840	145.992608	1351.90952	144.5978	1331.281	3.660	54.23891	0.0000000	0.6301511
0.21441	0.05970681	145.492608	1320.92227	144.5545	1299.796	3.660	54.23891	0.0000000	0.6301511
0.21643	0.05985538	145.992608	1289.17483	144.5068	1267.514	3.660	54.23891	0.0000000	0.6301511
0.21800	0.05999870	145.492608	1256.61300	144.4539	1234.370	3.660	54.23891	0.0000000	0.6301511
0.22002	0.06014295	145.992608	1230.67897	144.4084	1207.952	3.660	54.23891	0.0000000	0.6301511
0.22205	0.06028739	145.492608	1194.21187	144.3437	1172.806	3.660	54.23891	0.0000000	0.6301511
0.22407	0.06043202	145.992608	1160.67863	144.2785	1136.529	3.660	54.23891	0.0000000	0.6301511
0.22610	0.06057684	145.492608	1123.97137	144.1871	1099.001	3.660	54.23891	0.0000000	0.6301511
0.22812	0.06072185	145.992608	1085.74689	144.0987	1059.858	3.660	54.23891	0.0000000	0.6301511
0.23014	0.06086706	145.492608	1046.03785	143.9788	1019.119	3.660	54.23891	0.0000000	0.6301511

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0.23217	0.10479595	145.492608	1004.45232	143.8470	976.567	3.660	54.23891	0.0000000	0.4301511
0.23419	0.19354111	145.492608	961.35276	143.6895	931.931	3.660	54.23891	0.0000000	0.4301511
0.23622	0.20370017	145.492608	915.83922	143.4979	884.862	3.660	54.23891	0.0000000	0.4301511
0.23824	0.21570009	145.492608	867.72364	143.2595	834.907	3.660	54.23891	0.0000000	0.4301511
0.24026	0.23022073	145.492608	816.48883	143.9544	781.445	3.660	54.23891	0.0000000	0.4301511
0.24229	0.24827182	145.492608	761.41943	142.5491	723.401	3.660	54.23891	0.0000000	0.4301511
0.24431	0.27163139	145.492608	701.47791	141.9831	660.059	3.660	54.23891	0.0000000	0.4301511
0.24634	0.30364898	145.492608	635.06363	141.1333	588.491	3.660	54.23891	0.0000000	0.4301511
0.24836	0.35172307	145.492608	559.50369	139.7030	505.646	3.660	54.23891	0.0000000	0.4301511
0.25038	0.43696755	145.492608	469.96922	136.7457	402.672	3.660	54.23891	0.0000000	0.4301511
0.25241	0.66028718	145.492608	360.66127	126.9510	256.761	3.660	54.23891	0.0000000	0.4301511
0.25300	1.00000000	145.492608	322.76830	108.9832	157.881	3.660	54.23891	0.0000000	0.4301511

INTERATION= 1	MASS FLOW= 1.430000E-006
INTERATION= 2	MASS FLOW= 5.433333E-007
INTERATION= 3	MASS FLOW= 9.055556E-007
INTERATION= 4	MASS FLOW= 6.640741E-007
INTERATION= 5	MASS FLOW= 7.445679E-007
INTERATION= 6	MASS FLOW= 7.982305E-007
INTERATION= 7	MASS FLOW= 8.340055E-007
INTERATION= 8	MASS FLOW= 8.578555E-007

APPENDIX B: CALCULATION OF INLET MACH NUMBER

The stagnation conditions of the dewar are equivalent to the entrance stagnation conditions. The continuity equation for one-dimensional flow states that

$$\dot{m} = \rho VA. \quad (B.1)$$

The definition of Mach number is

$$M = V/[(\gamma RT)^{1/2}] \quad (B.2)$$

Combined with (B.1), this yields

$$\dot{m} = (\gamma RT)^{1/2} \rho MA \quad (B.3)$$

The density may be expressed in terms of the stagnation density and the Mach number.

$$\rho = \rho_0 \{1 + [(\gamma - 1)/2] M^2\}^{-1/(\gamma-1)} \quad (B.4)$$

The ideal gas equation states

$$\rho_0 = P_0/RT_0 \quad (B.5)$$

Combining Eqs. (B.4) and (B.5) yields

$$\rho = [P_0/(RT_0)] \{1 + (\gamma - 1)/2 M^2\}^{-1/(\gamma-1)} \quad (B.6)$$

Similarly, T may be expressed in terms of M and T₀

$$T = T_0 \{1 + (\gamma - 1)/2 M^2\}^{-1} \quad (B.7)$$

Combining Eqs. (B.3), (B.6), and (B.7) yields

$$\dot{m} = (\gamma/RT_0) A M P_0 \{1 + [(\gamma - 1)/2] M^2\}^{[\gamma+1]/[2(1-\gamma)]} \quad (B.8)$$

We wish to solve for the Mach number. Since the equation is nonlinear in M, the Newton-Raphson method of solution is invoked. Starting from some initial condition, new values of M are calculated iteratively until convergence is obtained. Using the expression

$$M_{\text{new}} = M_{\text{old}} - [f(M_{\text{old}})]/[f'(M_{\text{old}})] \quad (B.9)$$

where

$$f(M) = (\gamma/RT_0)^{1/2} A M P_0 \{1 + [(\gamma - 1)/2] M^2\}^{[\gamma+1]/[2(1-\gamma)]} - \dot{m} \quad (B.10)$$

and

$$f'(M) = AP_0 (\gamma/RT_0)^{1/2} B^{[(\gamma+1)/2][(1-\gamma)]} [1 - M^2[(\gamma+1)/2] B^{-1}] \quad (B.11)$$

B is an often used function of M and γ .

$$B = 1 + [(\gamma - 1)/2] M^2 \quad (B.12)$$

APPENDIX C: FRICTION FLOW MACH NUMBER CALCULATION

The differential equation relating the change in Mach number to the change in distance is

$$dM/M = \{ [\gamma M^2 B] / [2(1 - M^2)] \} [(4f dx)/D] \quad (C.1)$$

Solving with the imposed boundary conditions of $M = M_1$ at $x = x_1$ and $M = M_2$ at $x = x_2$ yields

$$\begin{aligned} [4f(x_2 - x_1)]/D = & \{ (M_2^2 - M_1^2) / (\gamma M_1^2 M_2^2) \\ & + [(\gamma + 1)/2\gamma] \ln[(M_1^2 B_2)/(M_2^2 B_1)] \} \end{aligned} \quad (C.2)$$

Solve for M_2 with the Newton-Raphson method using

$$\begin{aligned} f(M_2) = & [(M_2^2 - M_1^2) / (\gamma M_1^2 M_2^2)] \\ & + [(\gamma + 1)/2\gamma] \ln[(M_1^2 B_2)/(M_2^2 B_1)] - 4fL/D \end{aligned} \quad (C.3)$$

$$f'(M_2) = [2/(\gamma M_2^3) + \{ [(\gamma + 1) M_2] / \gamma \} [B_1^2 / (M_1^4 B_2^3)]] \quad (C.4)$$

where $L = x_2 - x_1$.

APPENDIX D: STAGNATION TEMPERATURE CALCULATION FROM AN ENERGY BALANCE

The heat transferred to a fluid as it travels through a pipe within isothermal walls is

$$\dot{m} dq = h(T_w - T_{aw}) \pi D dx \quad (D.1)$$

The differential change in heat of an ideal compressible gas is a function of the stagnation temperature.

$$dq = c_p dT_o \quad (D.2)$$

Combining Eqs. (D.1) and (D.2) yields

$$\dot{m} c_p dT_o = h(T_w - T_{aw}) \pi D dx \quad (D.3)$$

The adiabatic wall temperature of a laminar flow is equal to the stagnation temperature. For turbulent flow, the slope of the temperature profile of a fully developed fluid is too steep to apply a recovery factor; therefore, the adiabatic wall temperature will be assumed to be also equal to the stagnation temperature for turbulent fluid. Equation (D.3) may then be expressed as

$$\dot{m} c_p dT_o = h(T_w - T_o) \pi D dx \quad (D.4)$$

One may recall that the Nusselt number is defined as

$$Nu = hD/k_f \quad (D.5)$$

Equation (D.4) may be then expressed as $\dot{m} c_p dT_o = \pi Nu k_f (T_w - T_o) dx$.

$$dT_o/(T_w - T_o) = (\pi Nu k_f)/(\dot{m} c_p) dx \quad (D.6)$$

Integrating Eq. (D.6) with the boundary conditions

$T_o = T_{o1}$ at $x = x_1$ and $T_o = T_{o2}$ at $x = x_2$ yields

$$T_{o2} = T_w - \{[T_w - T_{o1}]/[\exp(\pi Nu k_f L/\dot{m} c_p)]\} \quad (D.7)$$

The exit stagnation temperature of a length L of pipe is therefore a function of known quantities.

APPENDIX E: HEAT TRANSFER WITH FRICTION

The change in Mach number in a pipe is a function of the change in length and the change in stagnation temperature.

$$\frac{dM}{M} = \frac{\gamma M^2 B}{2(1-M^2)} \frac{4f dx}{D} + \frac{(1 + \gamma M^2) B}{2(1-M^2)} \frac{dT_o}{T_o} \quad (E.1)$$

From the energy balance in APPENDIX D, we have

$$\dot{m} C_p dT_o = \rho V \pi/4 D^2 C_p dT_o = h(T_w - T_o) \pi D dx \quad (E.2)$$

One may express these relations in terms of dimensionless parameters.

$$dT_o/(T_w - T_o) = [4Nu_D/Pr(Re_D)] dx/D \quad (E.3)$$

For laminar flow,

$$Re_D = 16/f \quad (E.4)$$

$$[dT_o/(T_w - T_o)] = [4 Nu_D]/[Pr(16/f)] dx/D \quad (E.5)$$

$$(4f dx/D) = (16 Pr/Nu_D)[dT_o/(T_w - T_o)] \quad (E.6)$$

For laminar flow the differential equation is then,

$$\frac{dM}{M} = \frac{\gamma M^2 B}{2(1-M^2)} \frac{16 Pr}{Nu_D} \frac{dT_o}{T_w - T_o} + \frac{(1 + \gamma M^2) B}{2(1-M^2)} \frac{dT_o}{T_o} \quad (E.7)$$

For turbulent flow, one may invoke the Reynolds analogy

$$St = f/2 \quad (E.8)$$

and

$$(E.9)$$

$$dT_o/(T_w - T_o) = 4(f/2) dx/D \quad (E.10)$$

So that for turbulent flow,

$$\frac{dM}{M} = \frac{\gamma M^2 B}{2(1-M^2)} \frac{2 dT_o}{T_w - T_o} + \frac{(1 + \gamma M^2) B}{2(1-M^2)} \frac{dT_o}{T_o} \quad (E.11)$$

Equations (E.7) and (E.11) are solved numerically with the Runge-Kutta method, using about four iterations, as shown in APPENDIX F.

APPENDIX F: RUNGE-KUTTA SOLUTION

The basic Runge-Kutta equation is rather simple.

$$M_{n+1} = M_n + \Delta T_o / 6 (c_1 + 2c_2 + 2c_3 + c_4) \quad (F.1)$$

where

$$\begin{aligned} c_1 &= F(T_{on}, M_n) \\ c_2 &= F(T_{on} + \Delta T_o / 2, M_n + c_1 \Delta T_o / 2) \\ c_3 &= F(T_{on} + \Delta T_o / 2, M_n + c_2 \Delta T_o / 2) \\ c_4 &= F(T_{on} + \Delta T_o, M_n + c_3 \Delta T_o) \end{aligned} \quad (F.2)$$

The function F is simply a form of the differential equation we wish to solve for

$$F(T_{on}, M_n) = dM/dT_o|_n \quad (F.3)$$

For laminar flow,

$$F = \frac{dM}{dT_o} = \frac{MB}{2(1 - M^2)} \left[\gamma M^2 \left(\frac{16 Pr}{Nu_D (T_w - T_o)} \right) + \frac{1 + \gamma M^2}{T_o} \right] \quad (F.4)$$

For turbulent flow,

$$F = \frac{dM}{dT_o} = \frac{BM}{2(1 - M^2)} \left[\gamma M^2 \left(\frac{2}{T_w - T_o} \right) + \frac{1 + \gamma M^2}{T_o} \right] \quad (F.5)$$

APPENDIX G: FLUID PROPERTIES

All curve fits shown below were performed by a standard IMSL Math Library routine on the University of Illinois Cyber 175. They are valid only in a temperature range of 4.2 K to 350 K.

Thermal conductivity:

$$k_f = 0.0157218071 + 6.3416525 \times 10^{-4} T - 6.99407521 \times 10^{-7} T^2 + 4.52003096 \times 10^{-10} T^3 \quad (G.1)$$

Viscosity:

$$\mu = 1.10419265531 \times 10^{-6} + 1.160677594648 \times 10^{-7} T - 3.222922924062 \times 10^{-10} T^2 + 4.725700293922 \times 10^{-13} T^3 \quad (G.2)$$

Prandtl number:

$$Pr = 0.570232994 + 0.0055872291 T - 7.58324532 \times 10^{-5} T^2 + 4.32720501 \times 10^{-7} T^3 - 1.11661445 \times 10^{-9} T^4 + 1.07403542 \times 10^{-12} T^5 \quad (G.3)$$

APPENDIX H: MACH NUMBER CALCULATION WITH AREA CHANGE

In most texts, the area ratio is given in terms of the area at the throat, A^* . The following merely algebraically solves for some arbitrary A_2 .

$$A_1/A^* = \{1/M_1\} \{[2/(\gamma + 1)] B_1\}^{[\gamma+1]/[2(\gamma-1)]} \quad (H.1)$$

$$A_2/A^* = \{1/M_2\} \{[2/(\gamma + 1)] B_2\}^{[\gamma+1]/[2(\gamma-1)]} \quad (H.2)$$

$$A_2/A_1 = \{M_1/M_2\} (B_2/B_1)^{[\gamma+1]/[2(\gamma-1)]} \quad (H.3)$$

Solve for M_2 with a Newton-Raphson method.

$$f(M_2) = \{M_1/M_2\} (B_2/B_1)^{[\gamma+1]/[2(\gamma-1)]} - A_2/A_1 \quad (H.4)$$

$$f'(M_2) = M_1 \left(\frac{B_2}{B_1}\right)^{[\gamma+1]/[2(\gamma-1)]} \left[\frac{\gamma+1}{2} \left(\frac{B_1}{B_2}\right) - \frac{1}{M_2^2} \right] \quad (H.5)$$